

# Energy Efficiency (EE) e-training - East Africa

**Rahul Raju Dusa**  
Senior Expert

Monday, 22 March 2021

Copenhagen



COPENHAGEN CENTRE  
ON ENERGY EFFICIENCY  
SEforALL EE HUB

# Today's Agenda

#	Minutes	Title	Description
1	30 min	Energy Efficiency – HVAC Systems	Rahul Raju Dusa
2	20 min	Energy Efficiency – Lighting Systems	Clara Camarasa
3	20 min	Opportunities for Energy Efficiency in Water Supply Systems	Rahul Raju Dusa
4	20 min	Energy Efficiency in Water Supply Systems	Xianli Zhu
	10 min	Q & A	

# Energy Efficiency - HVAC

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What?

Why?

When?

# Energy Efficiency - HVAC

Where?

How?

# What ?

# What is HVAC system

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Heating Ventilation and Air Conditioning System

Heating and cooling equipment

Circulating pumps

Fans and blowers

Cooling towers

Air Handling Units

Pipelines, ducts and insulation

Others

# What is HVAC system

## Heating Ventilation and Air Conditioning system components

Mixed-air plenum  
and outdoor air  
control

Air filters

Supply fan

Exhaust fan

Outdoor air intake

Ducts

Terminal devices

Return air system

Heating and  
cooling coils

Self contained  
heating or cooling  
unit

Cooling tower

Boiler

Control system

Chiller units

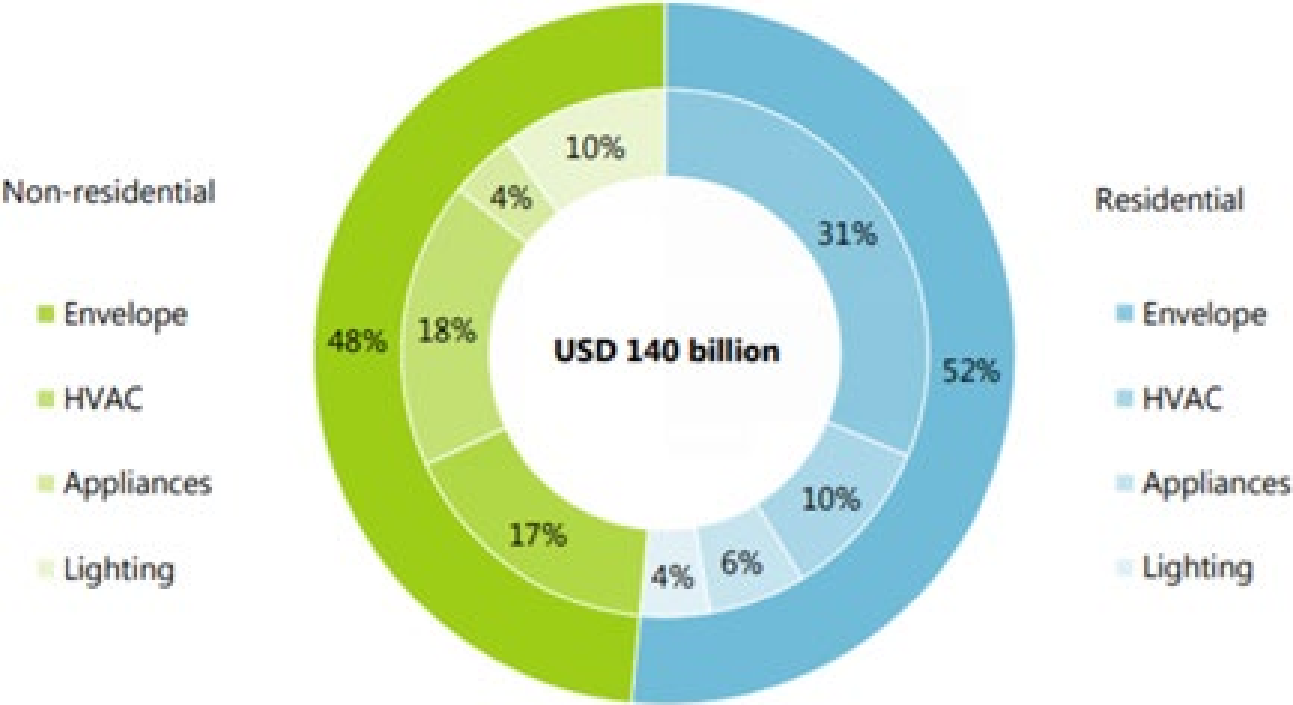
Humidification and  
dehumidification  
equipment

# Why ?



# Why HVAC system?

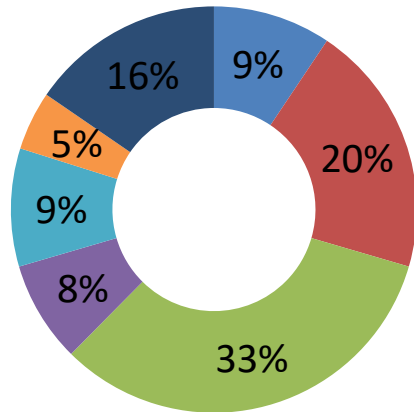
## Energy efficiency investment in buildings by subsector and end use, 2017



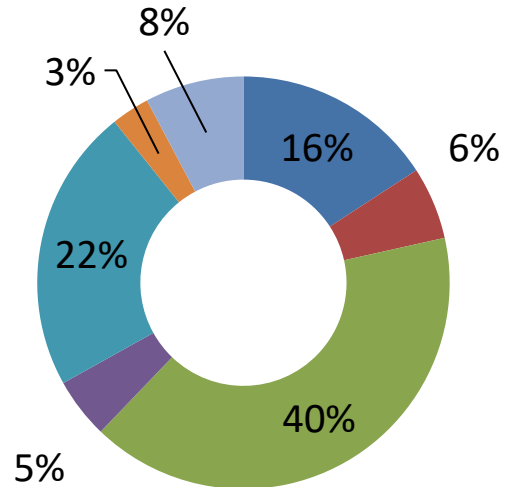
Source: IEA (2018), Energy Efficiency 2018, <https://www.iea.org/efficiency2018/>;

# Why HVAC?

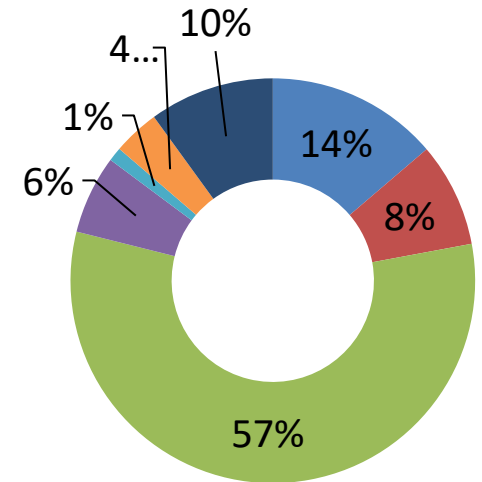
No. of proposals share



Share of energy savings



Share of energy cost savings



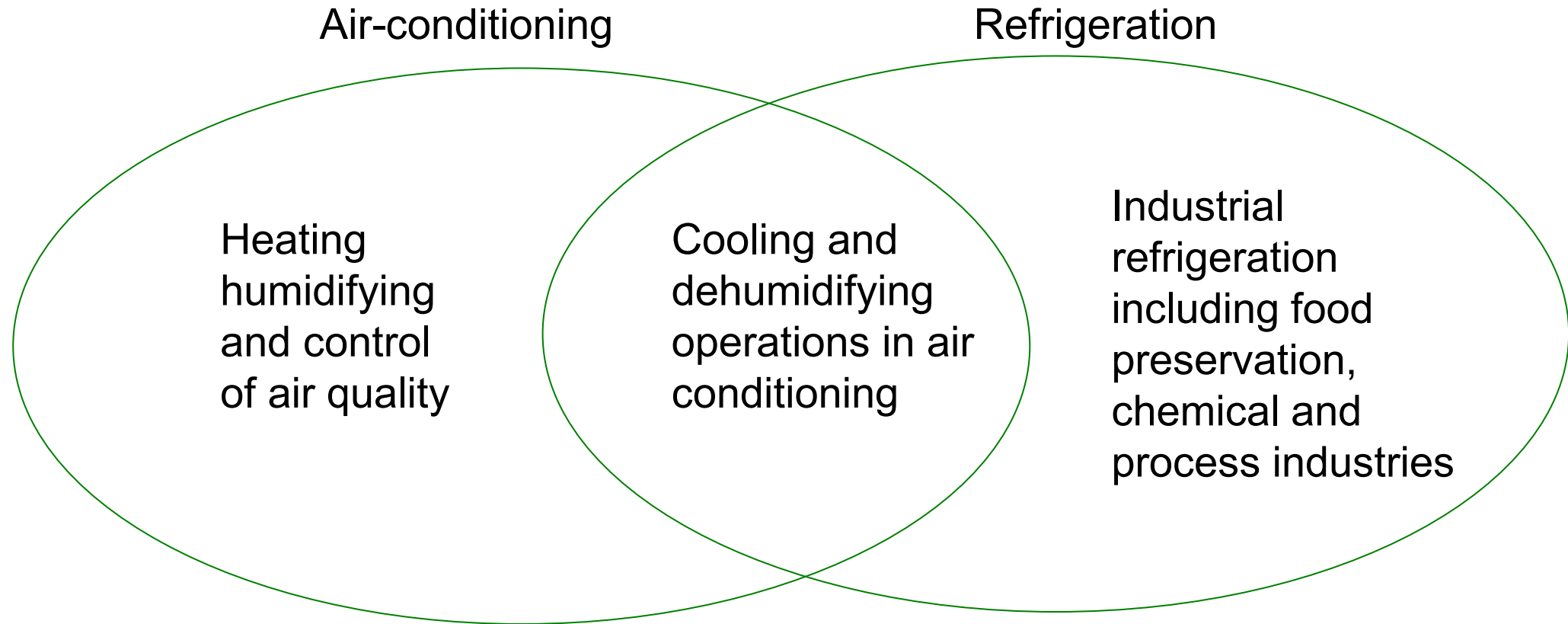
■ Electrical systems ■ Electrical drives ■ HVAC ■ Other Pumping and cooling towers ■ Boiler and steam ■ Compressed air ■ Lighting

\*Note: Representing climatic zones – Hot and Dry/Warm and Humid/Composite/Moderate/Tropical

Representing building types Educational, Institutional, Assembly Business, Industrial, Storage, Merchantile

# When and Where ?

# When and where do we need HVAC systems



Source: BEE India;

# When and where do we need HVAC systems

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Comfort air conditioning (20°C-25°C)

Process cooling – chilled water system (8°C-10°C)

Brine systems (sub-zero applications)

Source: BEE India;

# How ?

# How?

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How HVAC system works ?

How we assess energy efficiency performance ?

How to save energy ?

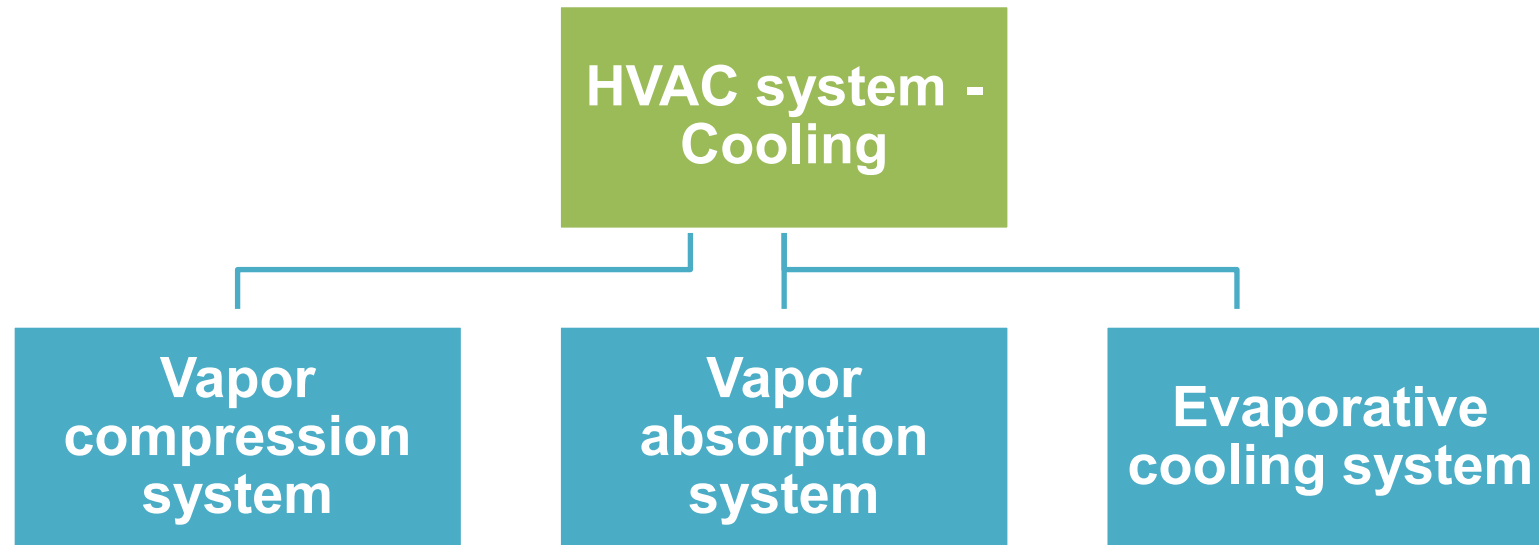
Source: BEE India;

# How HVAC system works?



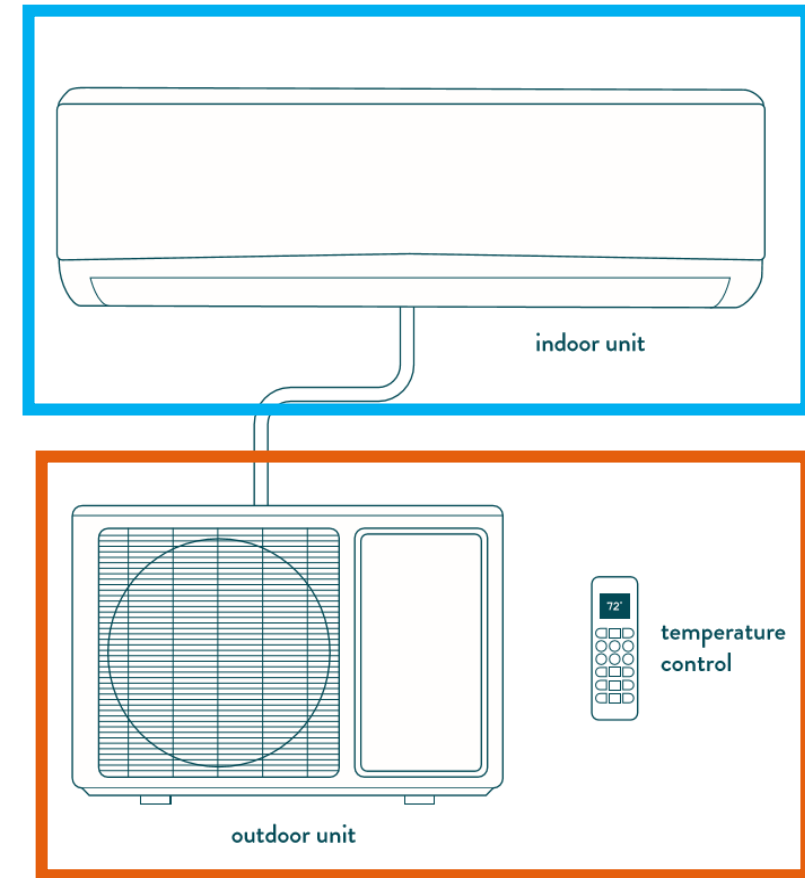
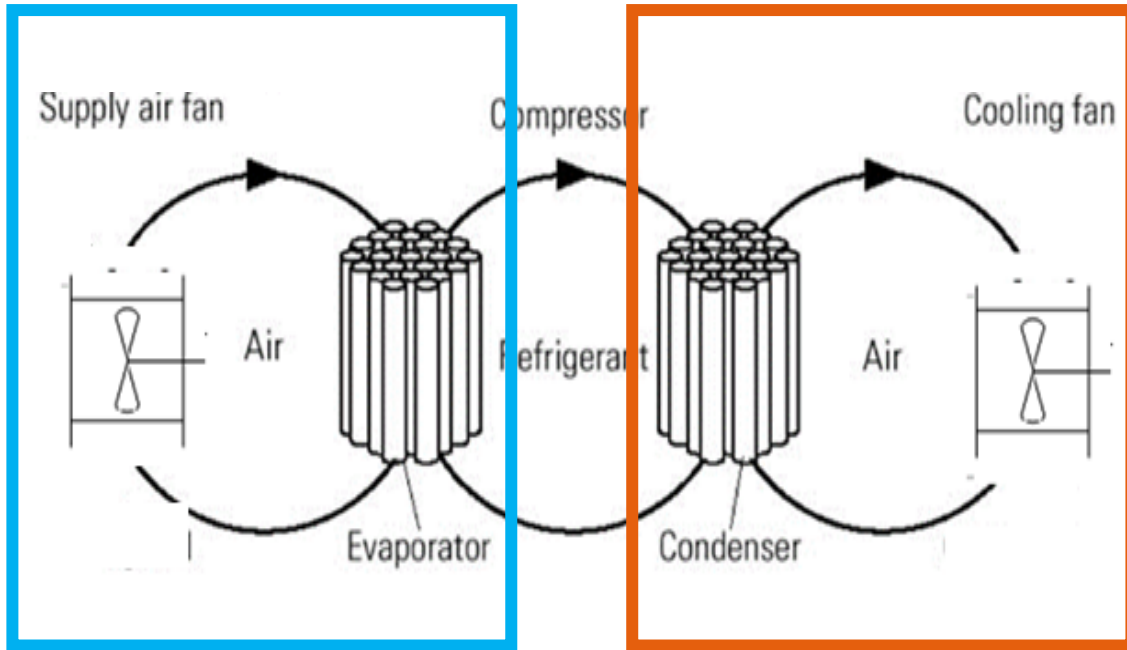
# Types – cooling and air conditioning

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# Vapour compression system – Heat transfer loops

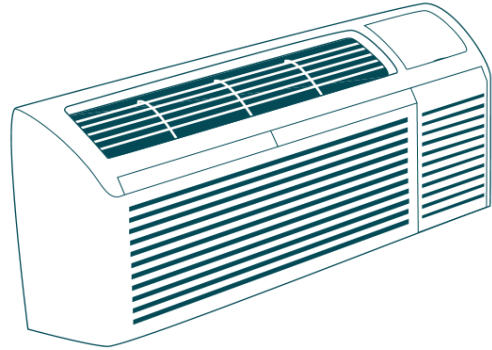
## Air cooled system (small capacities, D/X type)



Source: BEE India;

# Vapour compression system – Air cooled system (small capacities, D/X type)

Packaged terminal air-conditioner or heat pump

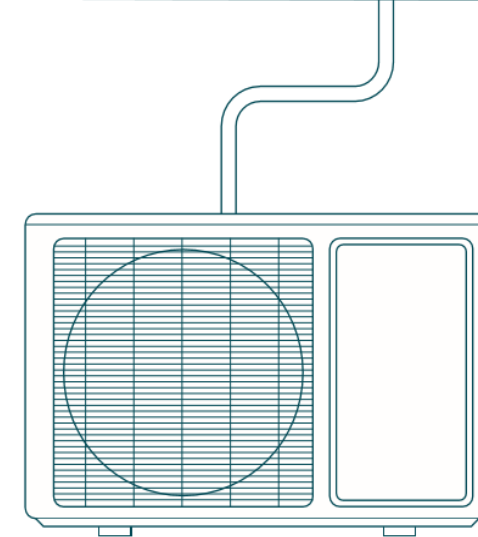
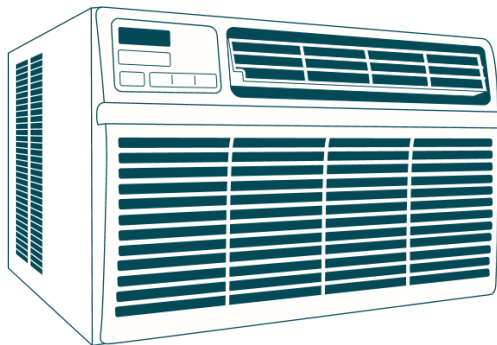


Non-ducted split system AC or heat pump



indoor unit

Window air-conditioner



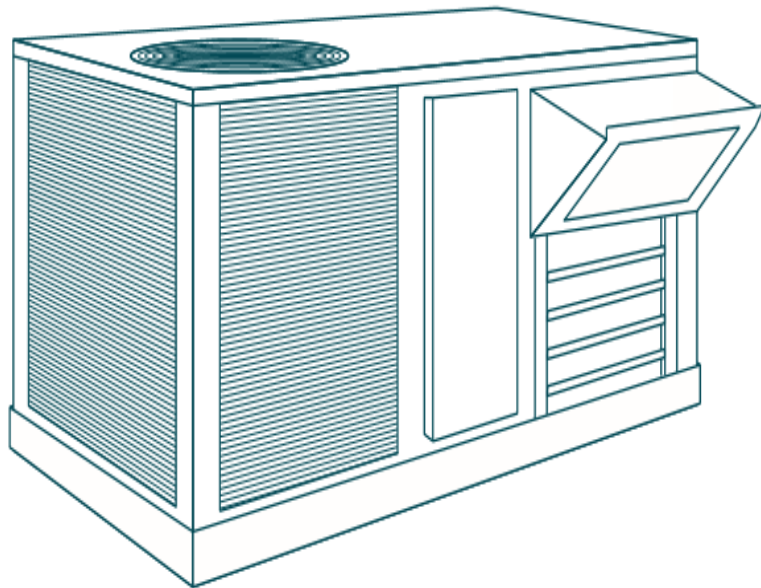
outdoor unit



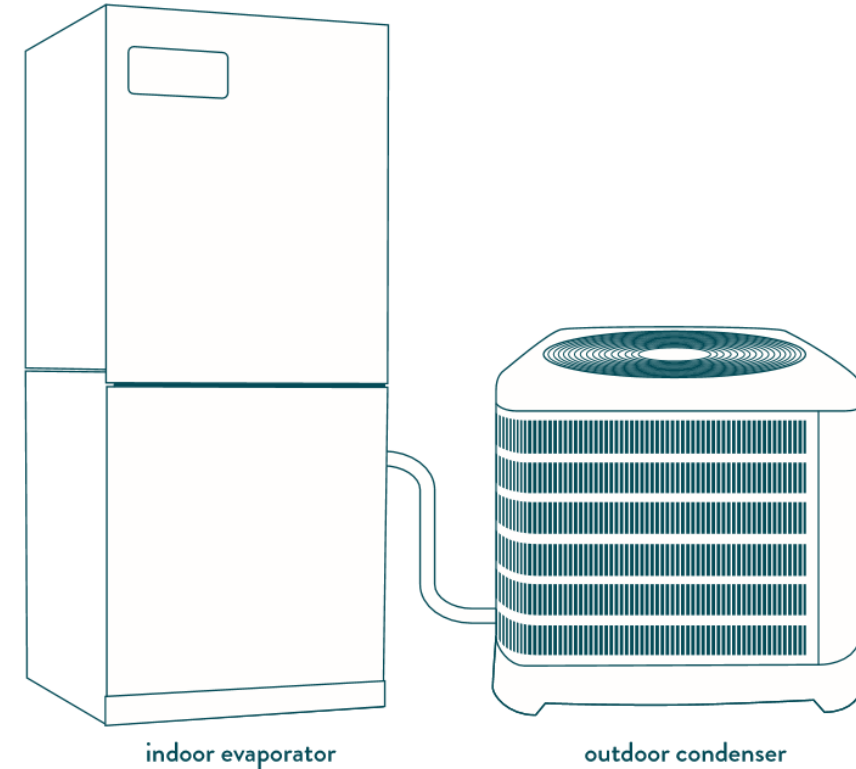
temperature control

# Vapour compression system – Air cooled system (small capacities, D/X type)

Packaged ducted AC or heat pump

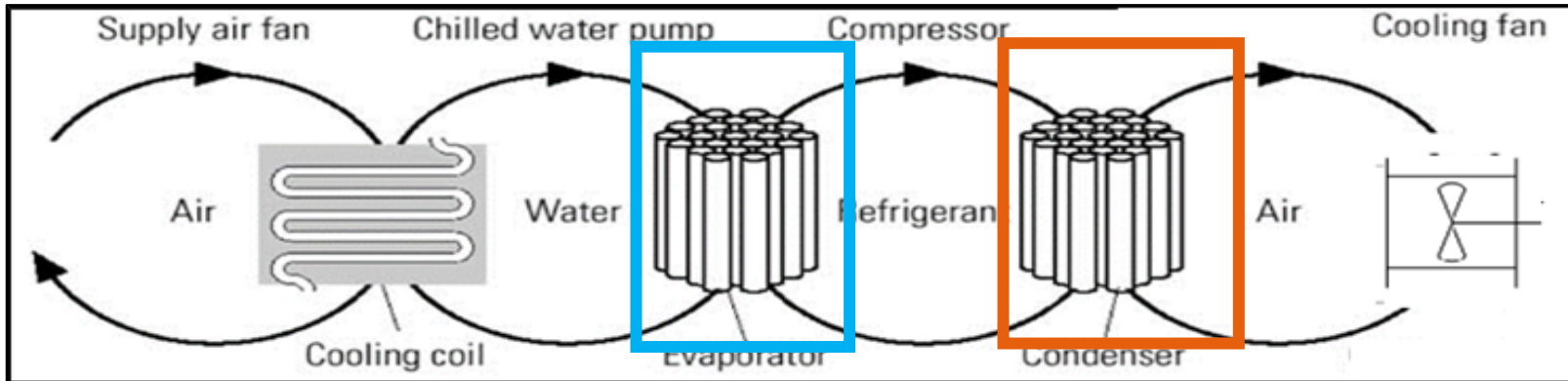


Ducted split system AC or heat pump



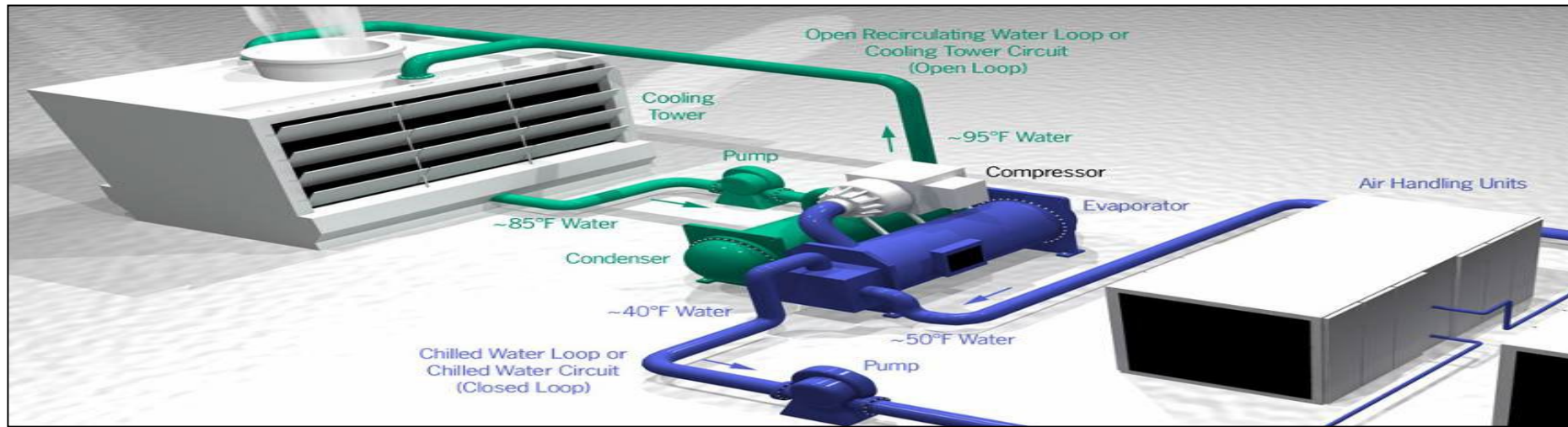
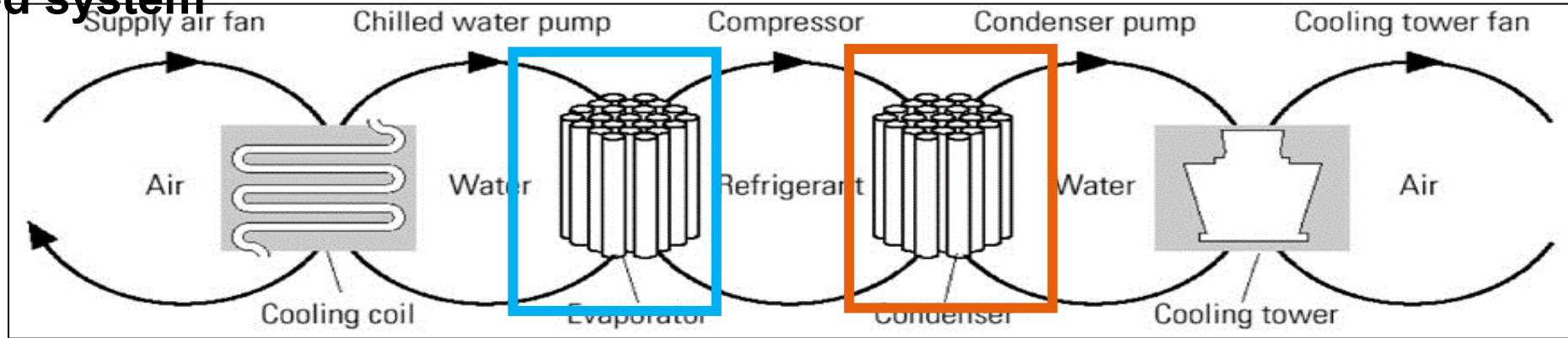
# Vapour compression system – Heat transfer loops

## Air cooled system



# Vapour compression system – Heat transfer loops

## Water cooled system

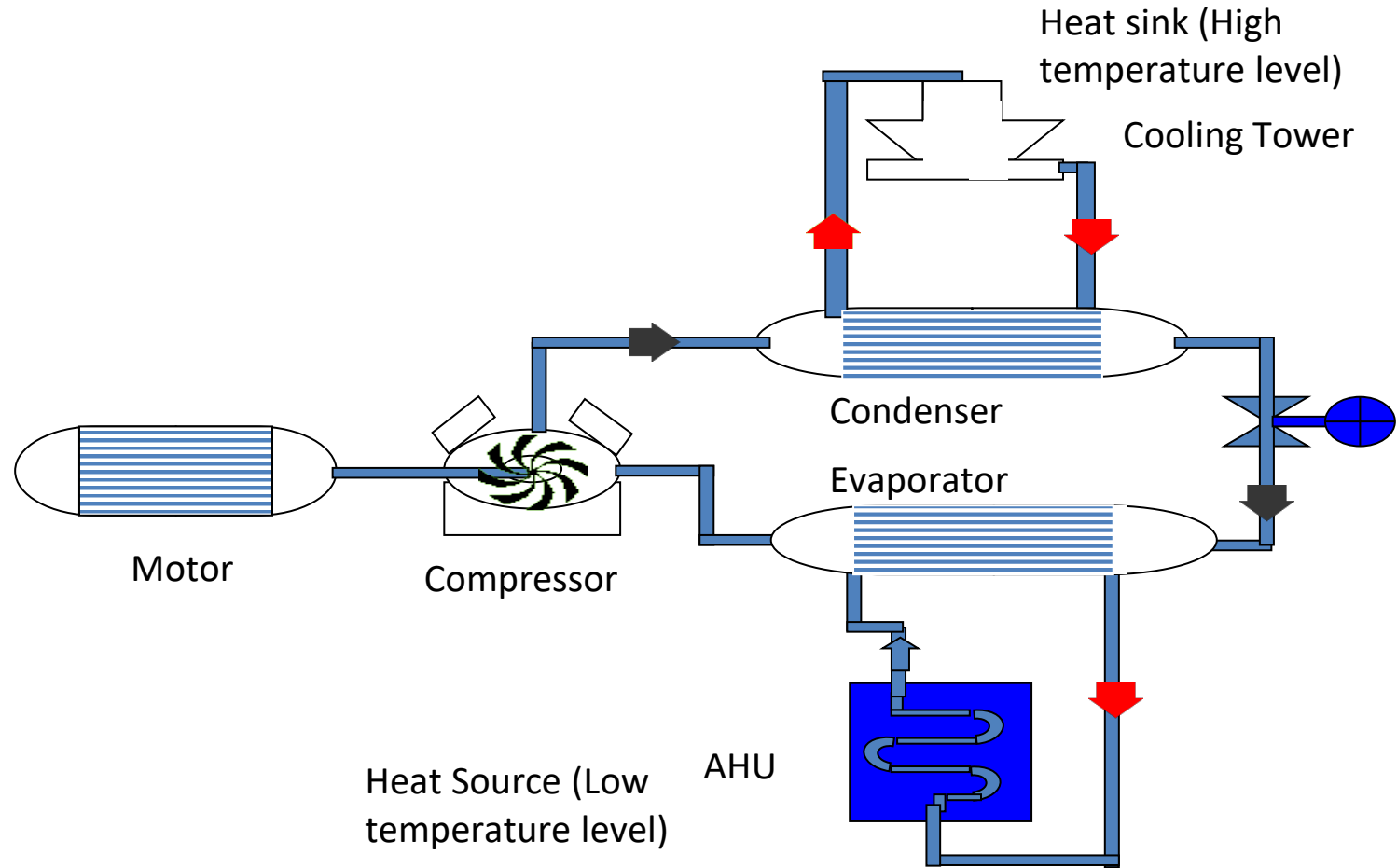
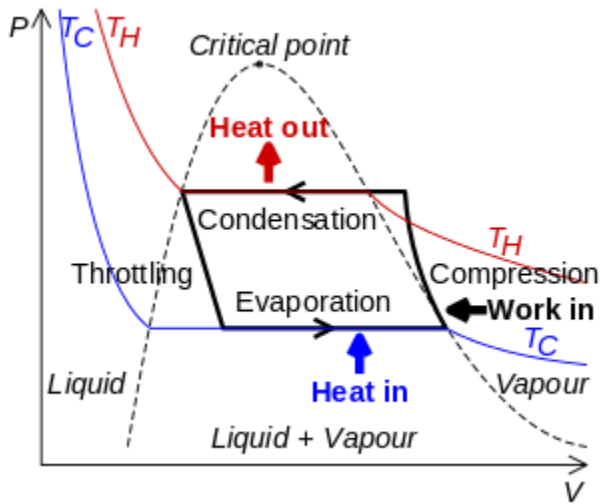


Source: 1. HVAC and Refrigeration system, Bureau of Energy Efficiency, India

2. Hudson Technologies  
UNEP DTU PARTNERSHIP  
COPENHAGEN CENTRE  
ON ENERGY EFFICIENCY  
SEforALL EE HUB



# Vapour compression system



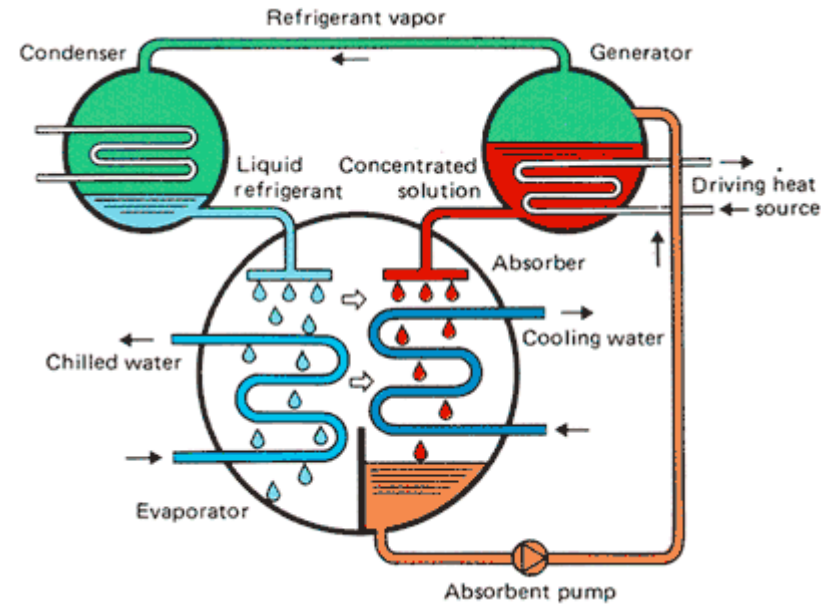
# Vapour compression system

## Comparison of different types of vapor compression systems

Parameters	Reciprocating	Centrifugal	Screw
1.Refrigeration temp. Range (Brine/water)	+7 to -30°C	+7 to 0°C	+7 to -25°C
2.Sp. Power consumption • Air conditioning • Sub zero temp.	<ul style="list-style-type: none"> <li>▪ 0.7–0.9 kW/TR</li> <li>▪ 1.2–2.5 kW/TR</li> </ul>	<ul style="list-style-type: none"> <li>▪ 0.59–0.63 kW/TR</li> </ul>	<ul style="list-style-type: none"> <li>▪ 0.65–0.7 kW/TR</li> <li>▪ 1.25–2.5 kW/TR</li> </ul>
3.Refrigerant <a href="#">*(For complete list)</a>	R <sub>11</sub> , R <sub>123</sub> , R <sub>134A</sub> Ammonia	R <sub>22</sub> , R <sub>12</sub>	R <sub>22</sub> , R <sub>134A</sub> Ammonia
4. Typical single unit capacity • Air conditioning • Sub zero temp.	<ul style="list-style-type: none"> <li>▪ Upto 150 TR</li> <li>▪ 10-50 TR</li> </ul>	<ul style="list-style-type: none"> <li>▪ 250 TR &amp; above</li> </ul>	<ul style="list-style-type: none"> <li>▪ 50-250 TR</li> <li>▪ 50-200 TR</li> </ul>



# Vapour Absorption system



# Types of VAM

Parameter	Single	Double	Half	Triple	Single (Ammonia)
Refrigeration temp (°C)	Above 6°C				Upto -33°C
Energy input (Heat)	Steam/Hot Water/Hot Oil/Direct fired	Steam/Hot Water/Hot Oil/Direct fired	Hot Water	Steam/Hot Oil/Direct fired	Steam/Hot Water/Hot Oil
Min heat input temp. (°C )	85	130	55	190	85
Energy to TR ratio (kcal/TR)	5000	2575	7500	2000	4615
Refrigerant	Pure water				Pure Ammonia
Absorbent	Water-LiBr solution				Ammonia – LiBr solution
Air conditioning range (TR)	≥30	≥30	≥30	≥50	≥30

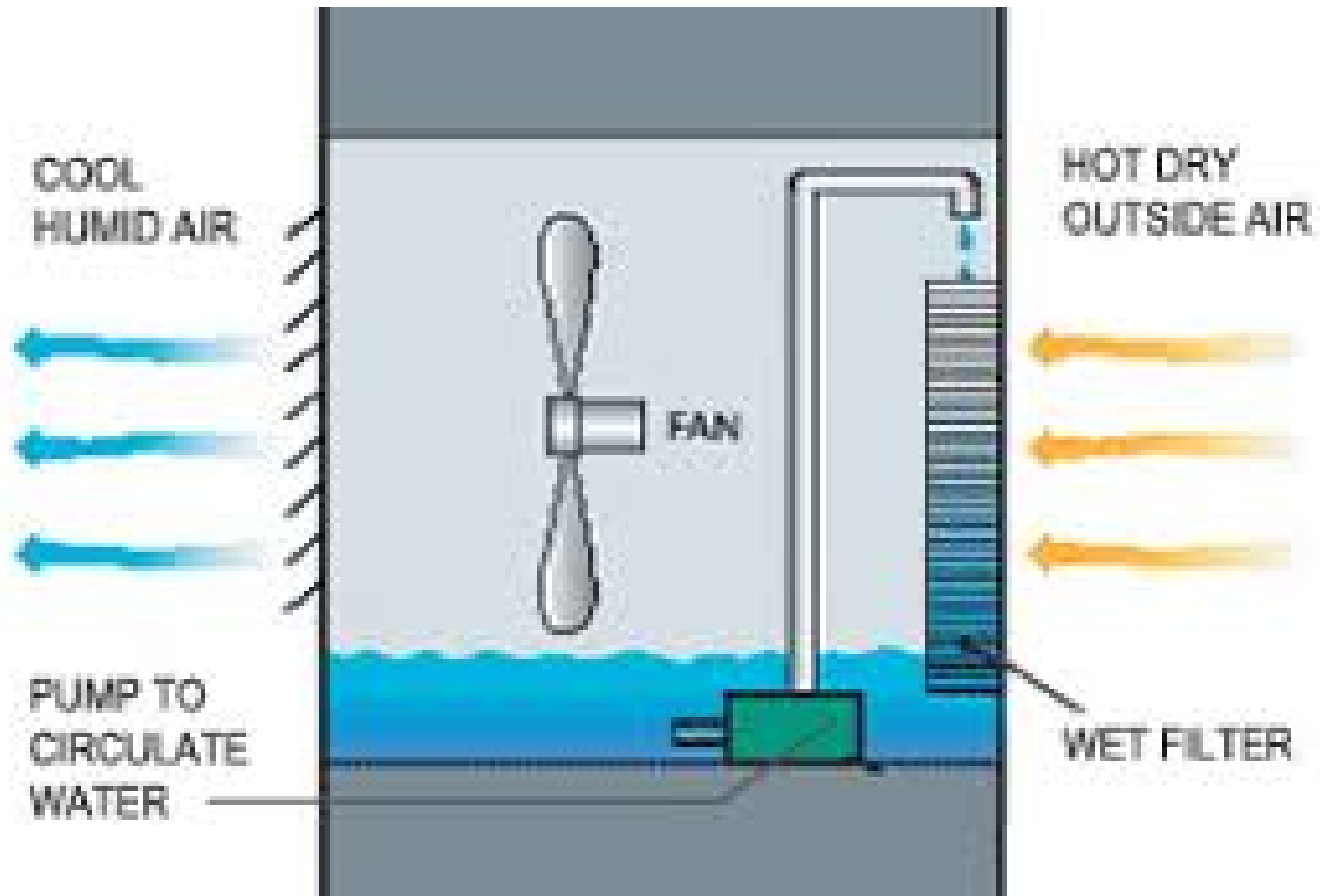
# VCM vs VAM

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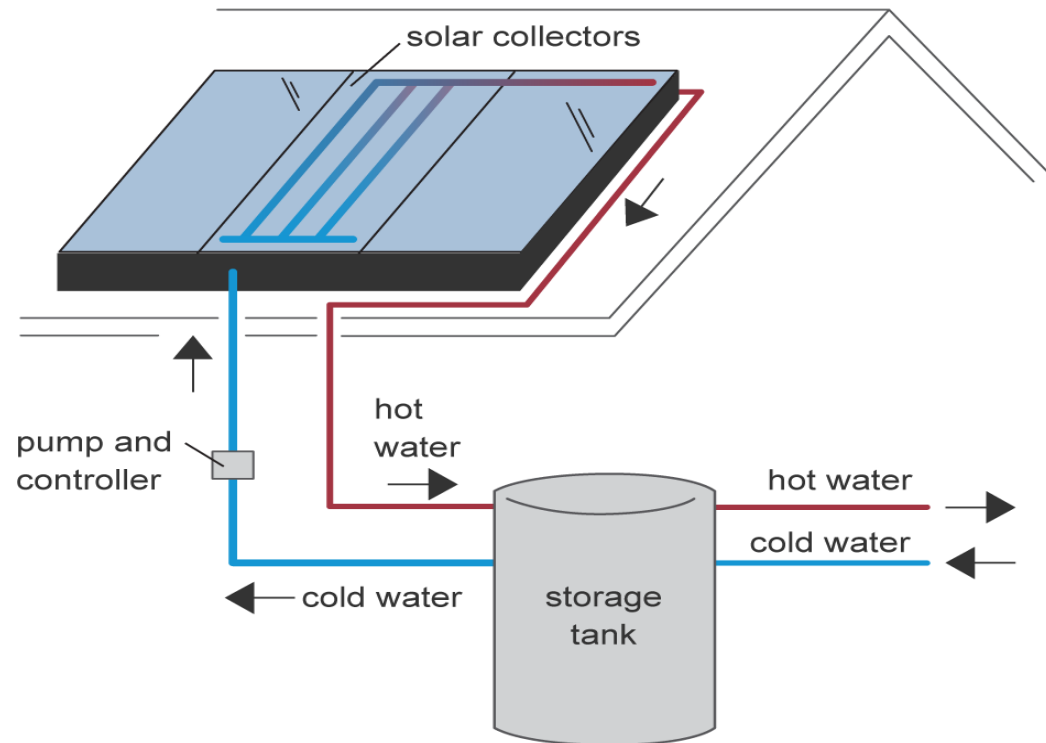
	VCM	VAM
Energy input	Electricity for compressor motor.	Electricity for two small circulating pumps only. Primary input heat energy
COP	4 to 5 for AC applications	Low 1.1 (for 2-stage LiBr machines)
Heat rejection factor	1.2	2.5 (very high – higher CT and pump capacity required)
Life span	Relatively high	Relatively low (LiBr is corrosive in nature)

**Use VAM only if you have waste heat or cheaper fuel source**

# Evaporative cooling



# Types - Heating



**Simple solar thermal system**

Source: <https://images.app.goo.gl/xswD85XiP4UNfFEY6>;

# Types - Heating

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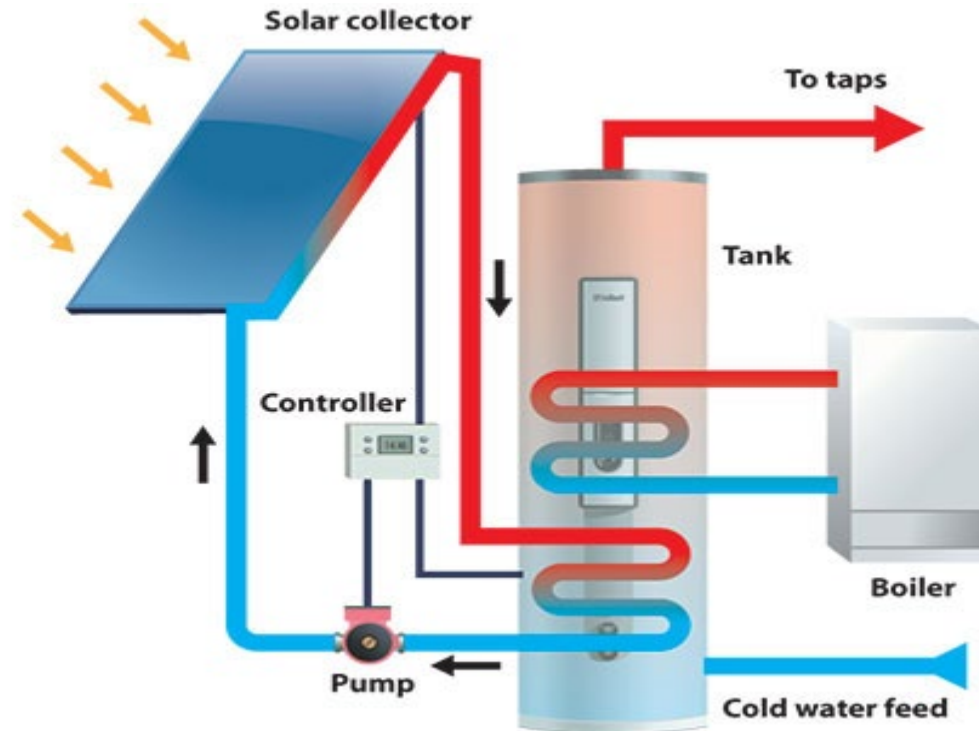


**Boilers (gas, electrical, oil/diesel, etc.)**

Source: <https://images.app.goo.gl/3QjZFGT8zZaF1Wv29>;

# Types - Heating

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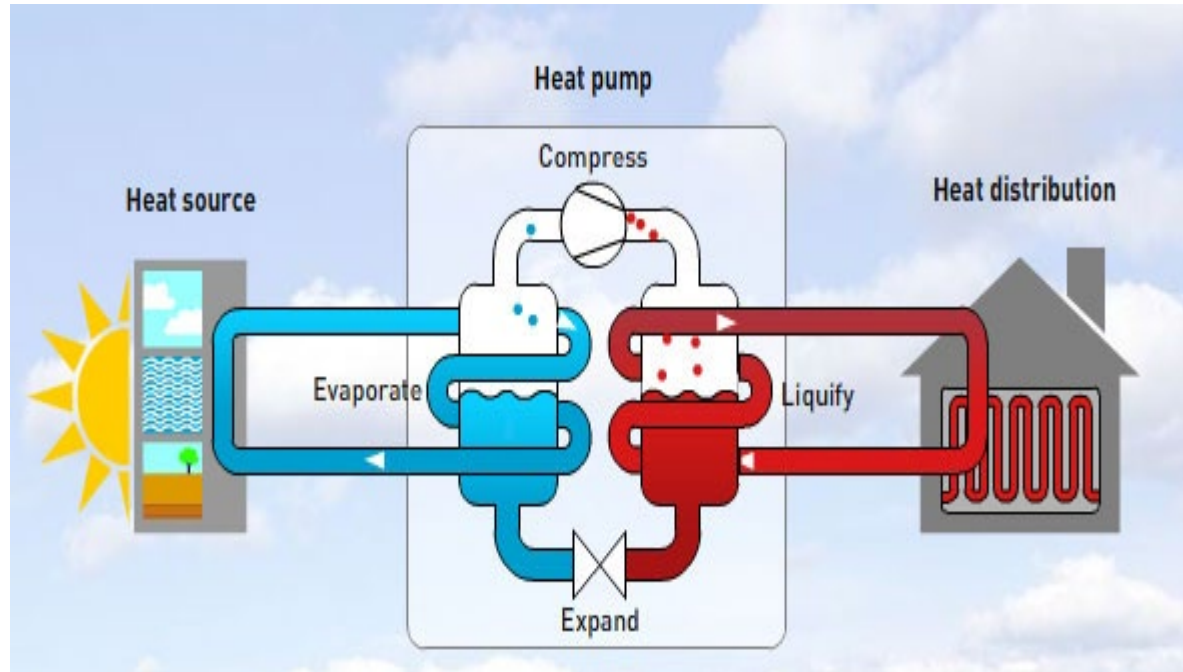


**Solar thermal + boiler integrated system**

Source: <https://images.app.goo.gl/bqnbZ3KJTxAVb1927>;



# Types - Heating



## Heat pumps

Source: <https://images.app.goo.gl/vDayAimQhbTCKdcD6>; <https://images.app.goo.gl/wkreCM6XqTJSeucX8>;



# Types - Heating

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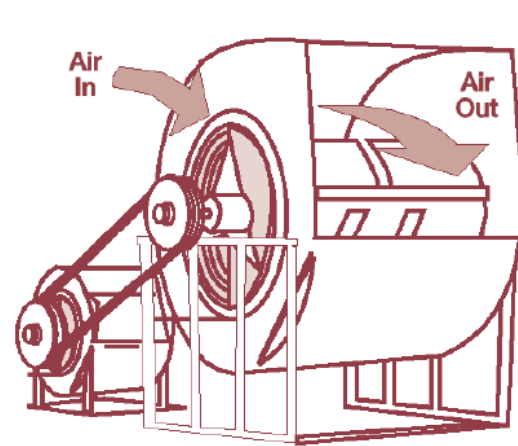


## Electrical resistance heaters

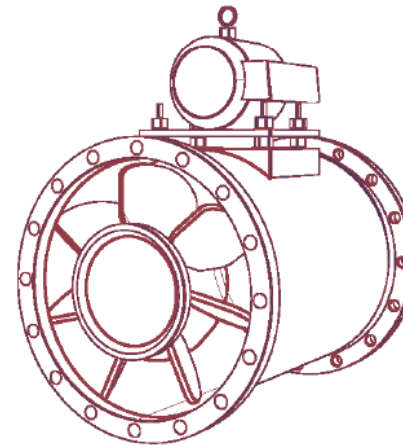
Source: <https://images.app.goo.gl/RrZHcrqm9A2uktWM7>; <https://images.app.goo.gl/C3q9LHN1FwBdoxKi6>;

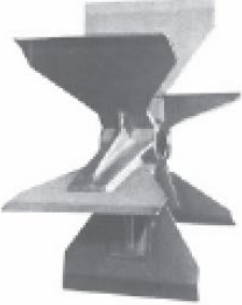
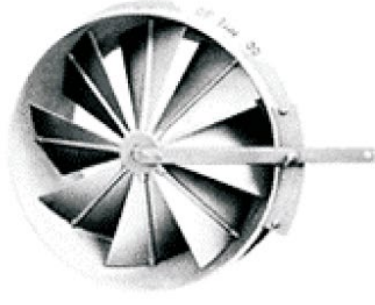

# Types - Ventilation




**Centrifugal**



**Axial**



Paddle Blade (Radial blade)	Forward Curved (Multi-Vane)	Backward Curved
		

Tube Axial	Vane Axial	Propeller
		

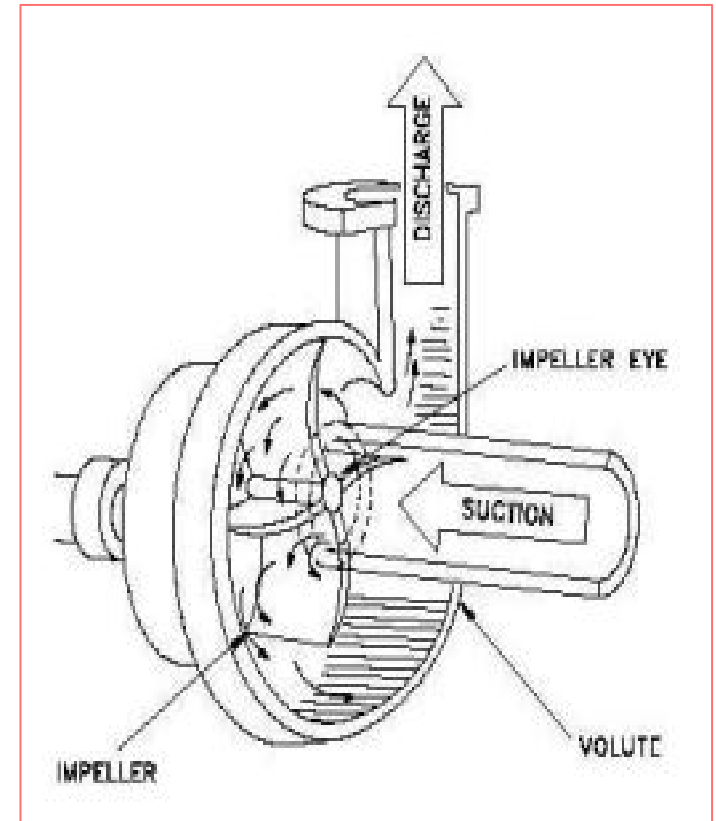
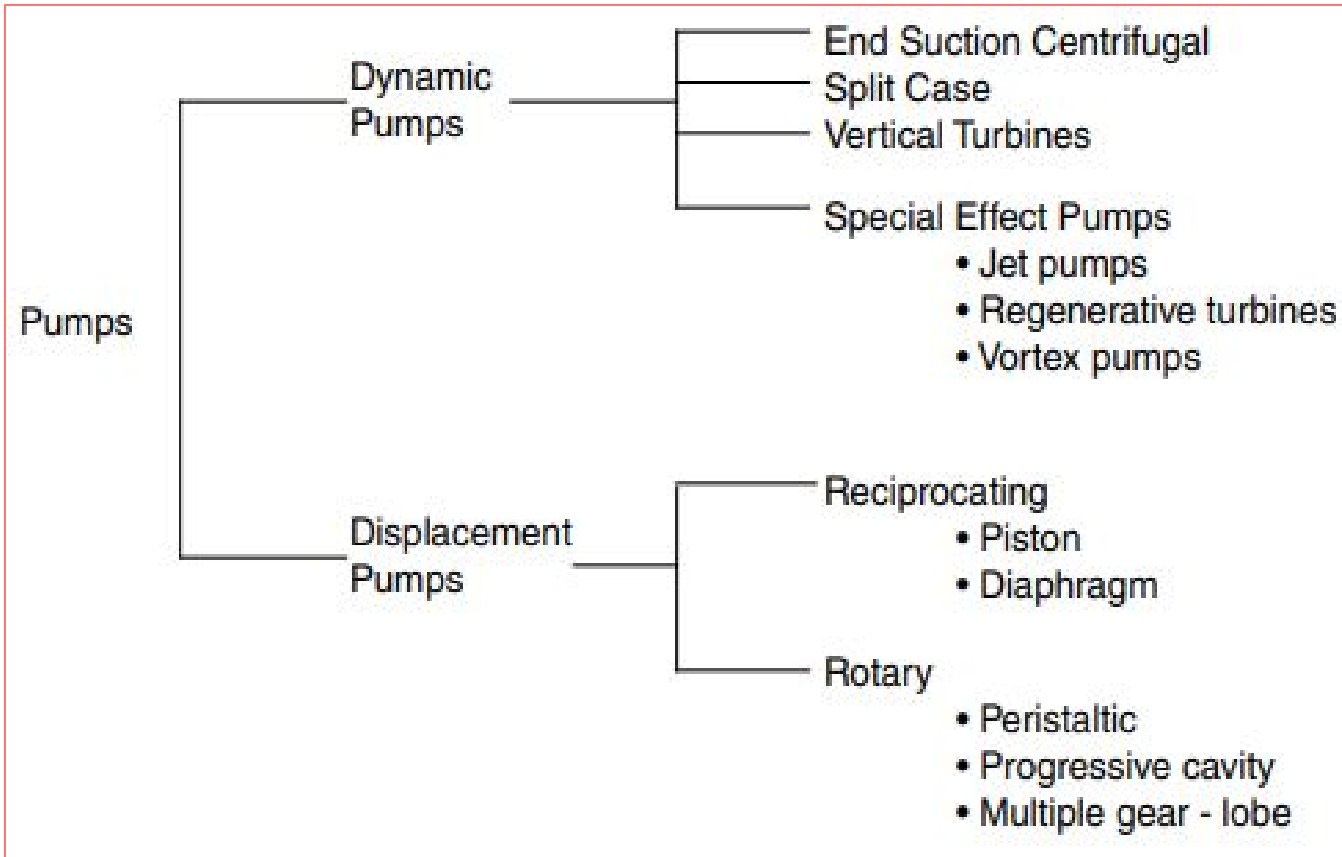
Source: BEE, India

# Types - Ventilation

Centrifugal			Axial		
Radial	High pressure / Medium flow	Industrial applications	Propeller	Low pressure, high flow	Air circulation, ventilation, exhaust
Forward curved	Medium pressure, high flow	Low pressure HVAC, packaged units, clean environment	Tube-axial	Medium pressure, high flow	HVAC, exhaust systems, ovens
Backward curved	High pressure, high flow	HVAC, industrial applications	Vane-axial	High pressure, medium flow	High pressure applications, HVAC systems

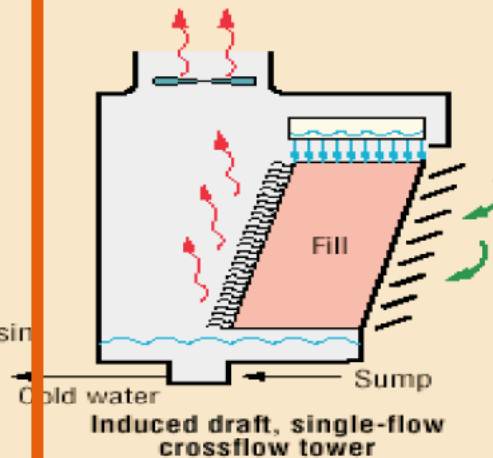
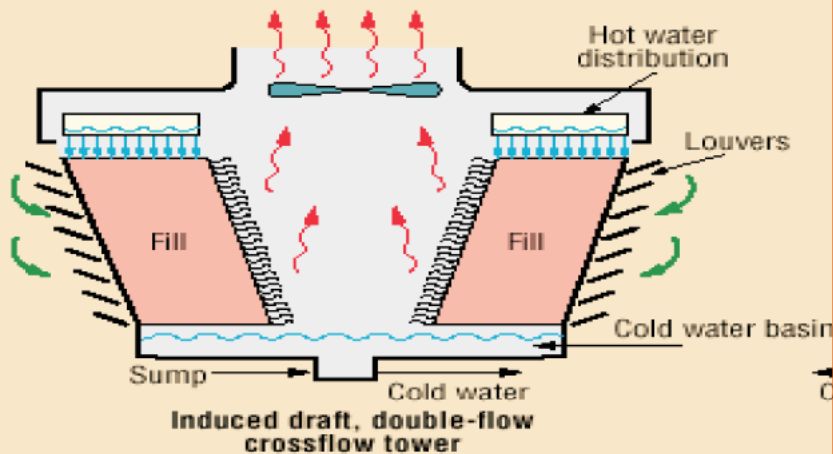
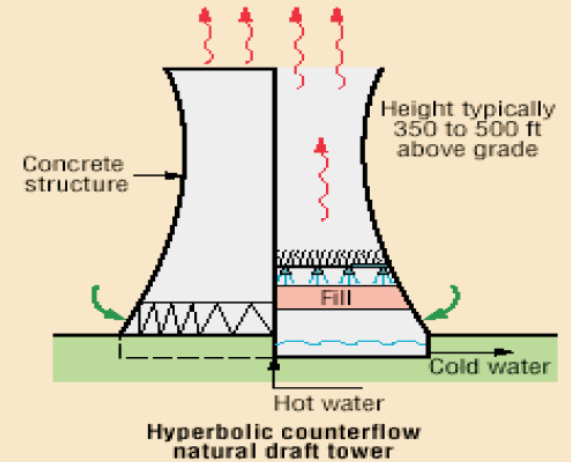
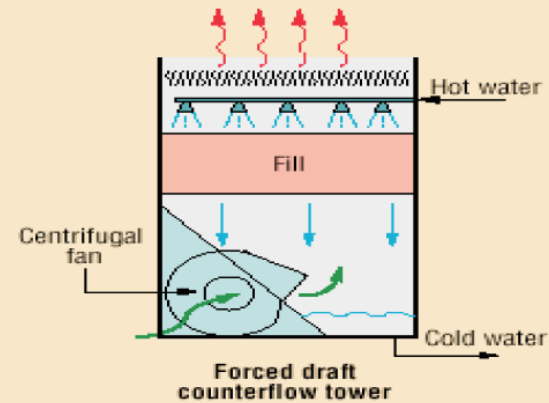
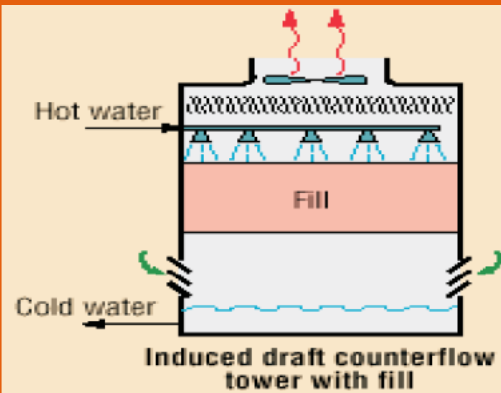
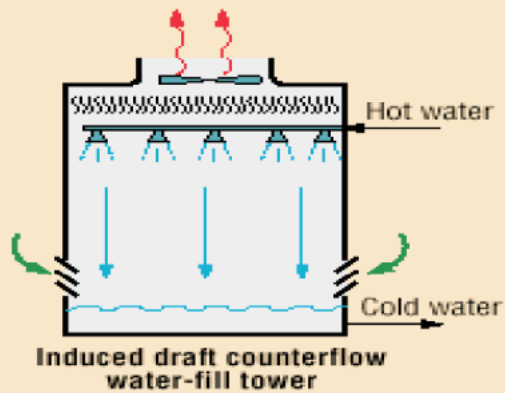
Source: BEE, India

# Types - Pumps



Source: BEE, India

# Types – Cooling towers



Source: BEE, India

# How we assess efficiency performance?



# Measurements of HVAC systems



# Chiller performance testing

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The cooling effect of refrigeration systems is measured in **Tonnes of Refrigeration (TR)**

A ton of refrigeration is defined as quantity of **heat to be removed** in order **to form 1 ton of ice in 24 hours** when the initial temperature of water is 0°C.

$$1 \text{ TR} = 3024 \text{ kcal/hour} = 3.51 \text{ kW} = 12,000 \text{ BTU/hour}$$



# Chiller performance testing

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## Efficiency ratios

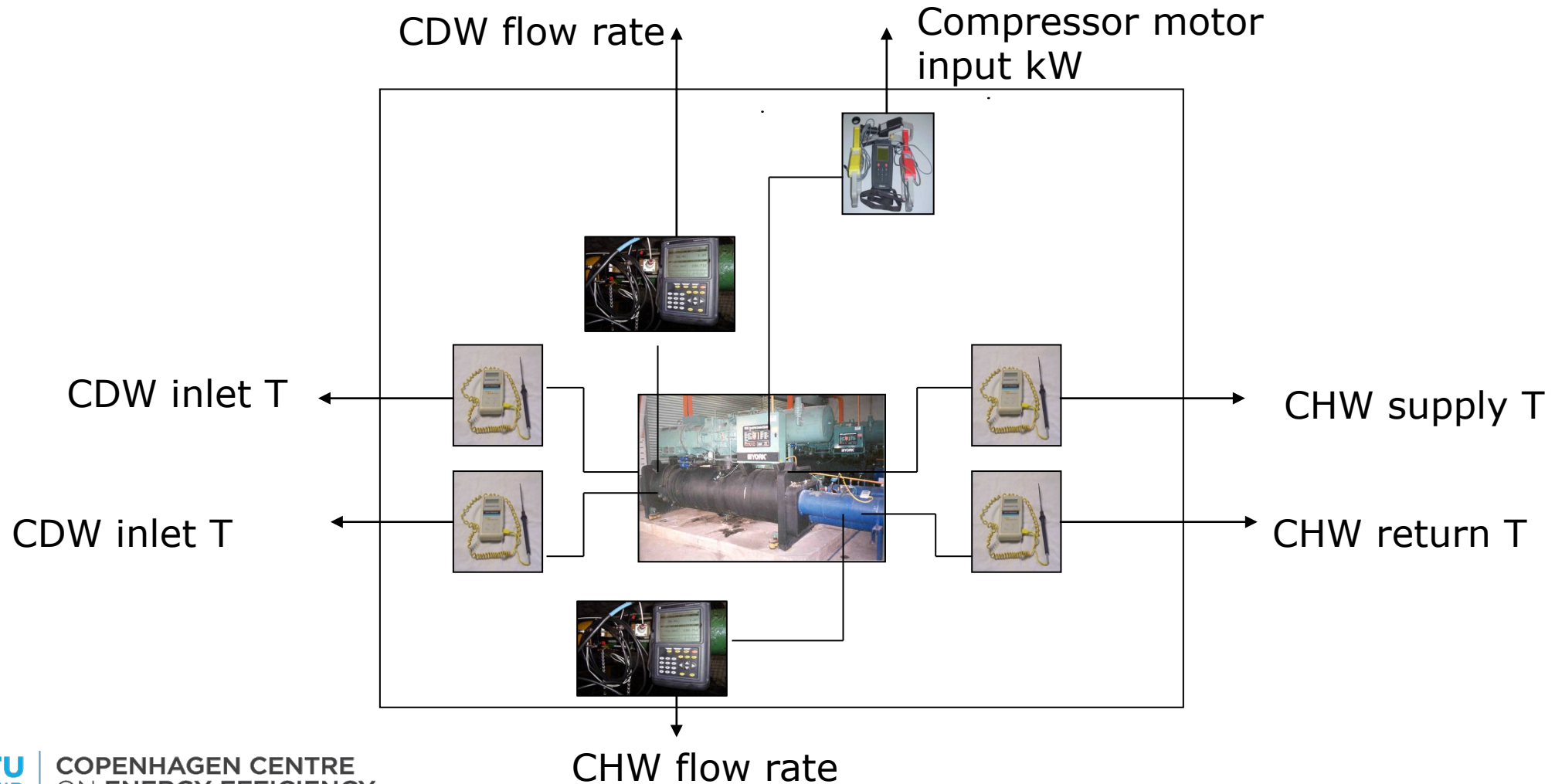
$$\text{Coefficient of Performance COP} = \frac{\text{Useful refrigerating effect or heat removed (kW)}}{\text{Net energy supply from external sources (kW)}}$$

$$\text{Energy Efficiency Ratio EER} = \frac{\text{Useful refrigerating effect } \left(\frac{\text{BTU}}{\text{hour}}\right)}{\text{Work done (W)}}$$

$$\text{Specific Power Consumption SPC} = \frac{\text{Power consumption (kW)}}{\text{Refrigeration effect (TR)}}$$

$$\text{kW/TR} = \frac{12}{\text{Energy Efficiency Ratio (EER)}} = \frac{3.516}{\text{Coefficient of Performance (COP)}}$$

# Chiller performance testing



# Chiller performance testing

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## Chiller

$$TR = \frac{\text{FlowRate} \left(\frac{\text{kg}}{\text{h}}\right) \times \text{SpecificHeat} \left(\frac{\text{kcal}}{\text{kg}^{\circ}\text{C}}\right) \times \text{Temp.difference} (^{\circ}\text{C})}{3024}$$

## Cooling coil

$$TR = \frac{\text{AirFlow} \left(\frac{\text{m}^3}{\text{h}}\right) \times \text{Density} \left(\frac{\text{kg}}{\text{m}^3}\right) \times \text{Enthalpy.difference} \left(\frac{\text{kcal}}{\text{kg}}\right)}{3024}$$

# Chiller performance testing

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- Screw chiller : 250 TR
- Chilled water flow rate : 42 lps
- Inlet water temperature : 12.2°C
- Outlet water temperature : 7.2°C
- Refrigeration capacity :  $\frac{42 \times 3600 \times 1 \times (12.2 - 7.2)}{3024} = 250 \text{ TR}$

Power consumption : 403.2 kW

Specific power consumption :  $\frac{403.2 \text{ kW}}{250 \text{ TR}} = 0.62 \frac{\text{kW}}{\text{TR}}$

# Chiller performance testing

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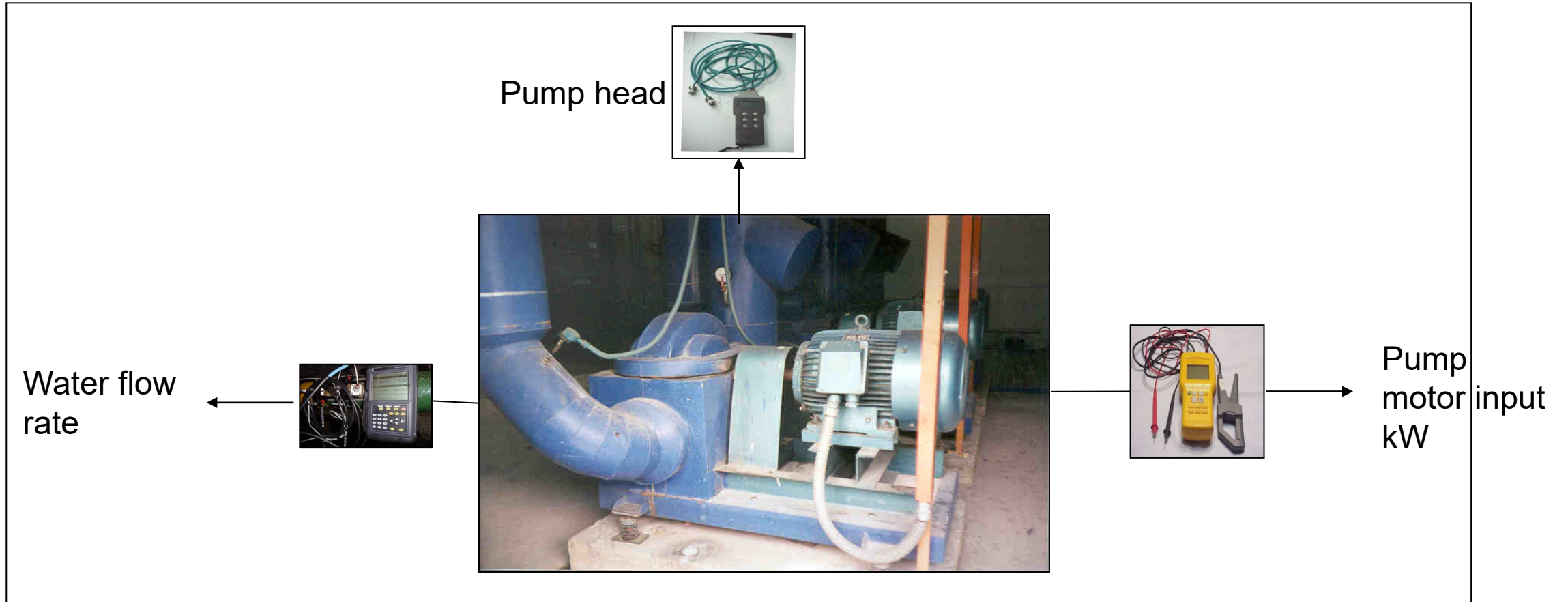
## Cooling load estimation

- Small office cabins - 0.1 TR/m<sup>2</sup>
- Medium size office - 0.06 TR/m<sup>2</sup>
- Large multi-storeyed office - 0.04 TR/m<sup>2</sup>

Note: for indicative purpose only. Cannot be taken as a basis for any investment grade activities.

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# Pump performance testing



# Pump performance testing

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## Pump efficiency

$$\text{Pump Efficiency} = \frac{\text{Hydraulic Power, } P_h}{\text{Power Input to the pump shaft}} \times 100$$

$$\text{Hydraulic Power, } P_h \text{ (kW)} = Q \times (H_d - H_s) \times \rho \times g / 1000$$

Q = Volume flow rate (m<sup>3</sup>/s)

ρ = Density of fluid (kg/m<sup>3</sup>)

g = Acceleration due to gravity (m/s<sup>2</sup>)

H<sub>d</sub> = Discharge pressure (meter)

H<sub>s</sub> = Suction pressure (meter)

$$\text{Power Input to the pump shaft} = \text{Drive input power} \times \text{Motor efficiency, \%}$$

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# Pump performance testing

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## Pump efficiency

### Pump operating parameters

Q = 350 m<sup>3</sup>/h, ρ = 1000 kg/m<sup>3</sup>, g = 9.81m/s<sup>2</sup>

Discharge pressure = 30 meter,

Suction pressure = 3 meter,

Total Head = (30 – 3) = 27 meter,

Input Power = 35kW

Motor efficiency = 92%

$$\text{Pump } \eta = \frac{\left(\frac{350}{3600}\right) \times 27 \times 1000 \times 9.81}{35 \times \left(\frac{92}{100}\right)} \times 100$$

$$\text{Pump } \eta = 80\%$$



# Pump performance testing

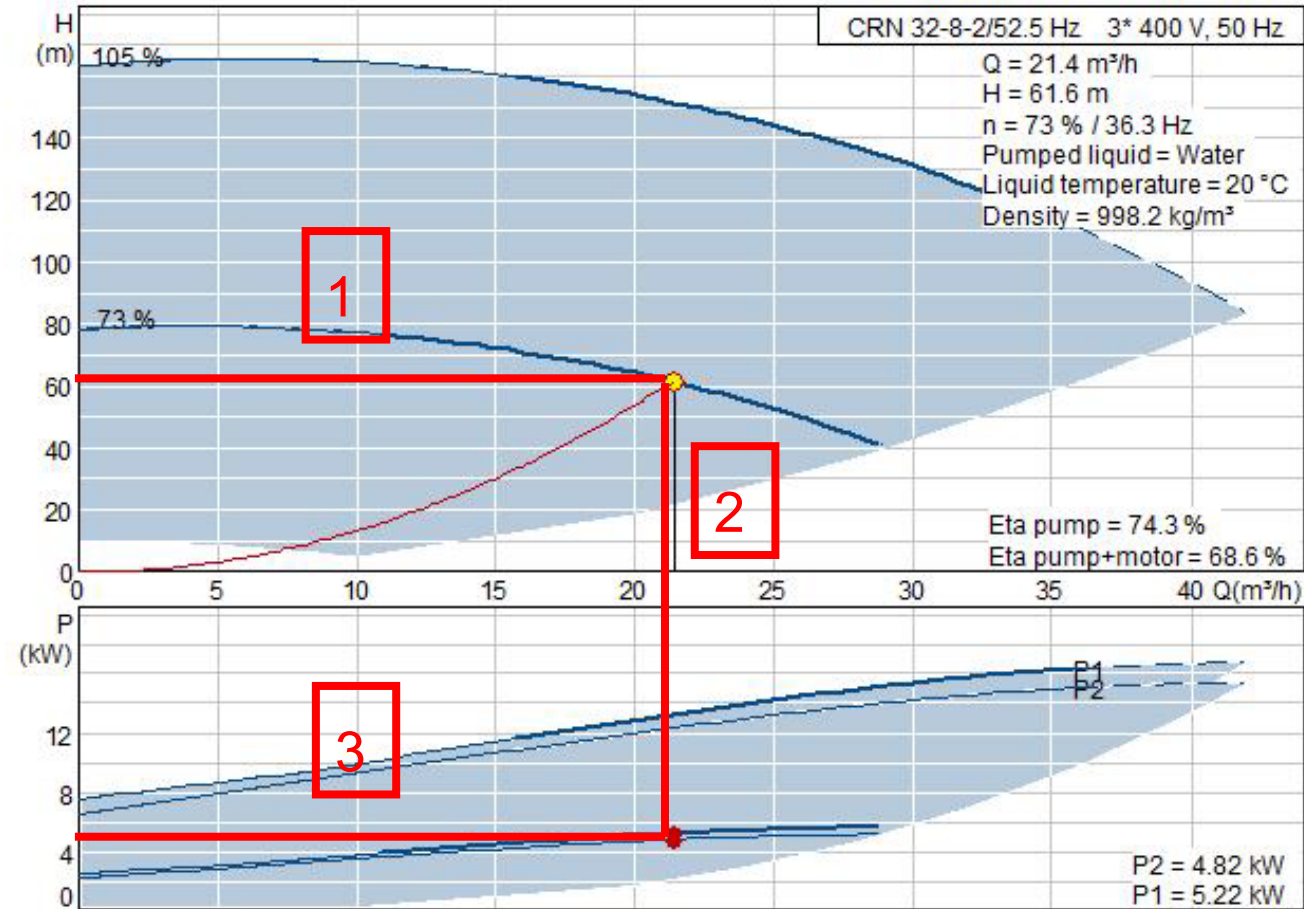
## Pump performance curve

$Q = 21.4 \text{ m}^3/\text{h}$ ,

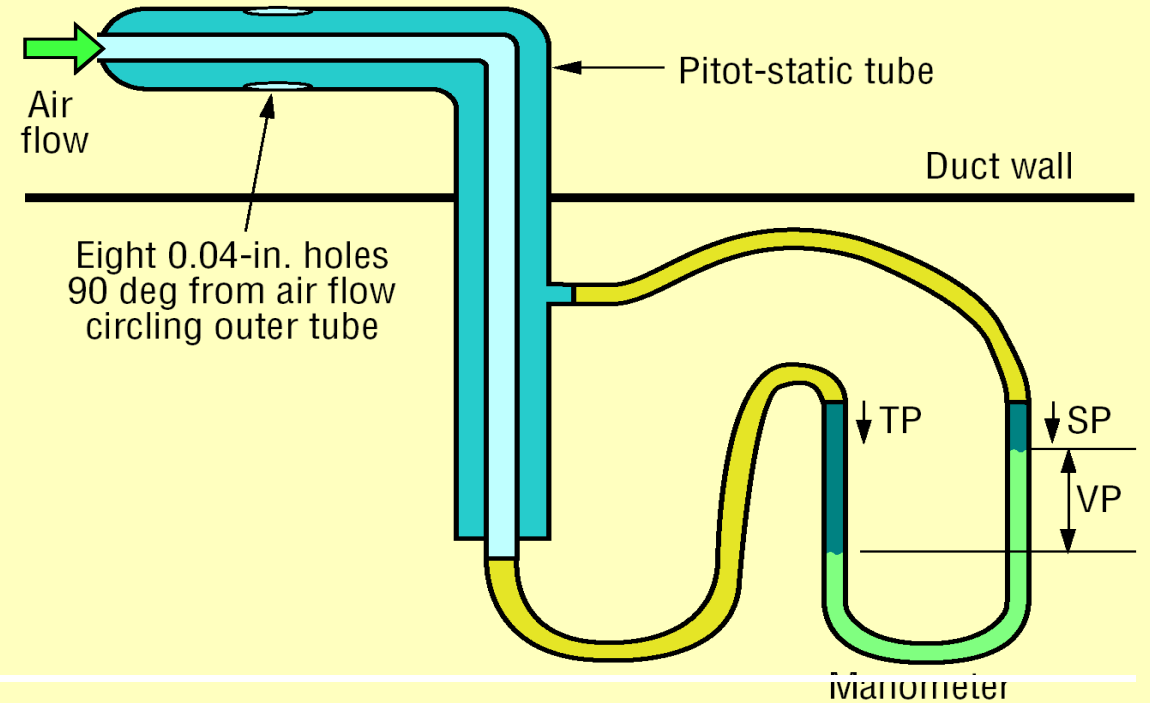
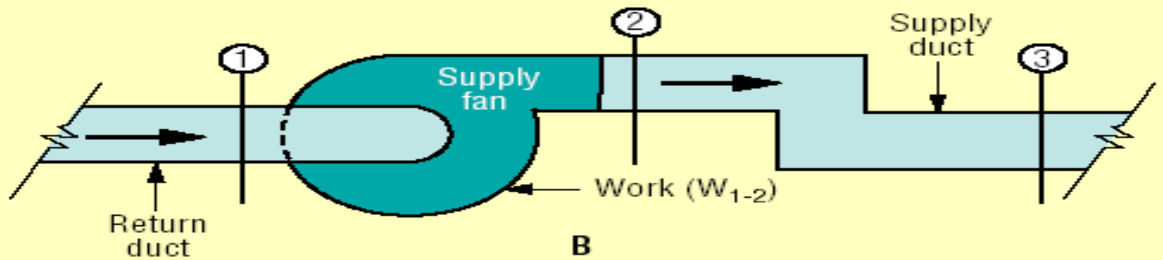
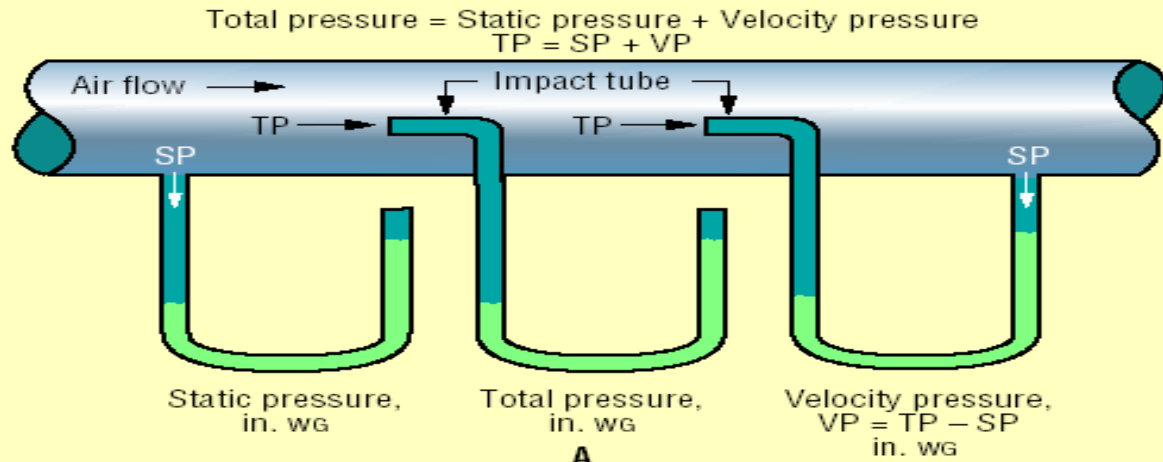
$H = 61.6 \text{ m}$

$P = 5.2 \text{ kW}$

Pump efficiency = 73%



# Fan & Blower performance testing



Source: BEE, India

# Fan & Blower performance testing

$$\rho_2 = \rho_1 \times \frac{T_1}{T_2} \times \frac{P_1}{P_2}$$

$\rho$ : Density, kg/m<sup>3</sup>

$P$ : Pressure, mm Wg (kg/m<sup>2</sup>)

$T$ : Temperature, Kelvin

Suffix -1 represents parameters at NTP

$\rho_1 = 1.29 \text{ kg/Nm}^3$

$P_1 = 1 \text{ bar} = 10330 \text{ mm wg}$

$T_1 = 0 \text{ }^\circ\text{C} = 273 \text{ K}$

Suffix – 2 represents measured parameters at sample point

$$\text{Velocity } V = C \times \frac{\sqrt{2 \times g \times h}}{\rho}$$

Where,

$C$ : Pitot factor

$g$ : Acceleration due to gravity, 9.81 m/s<sup>2</sup>

$h$ : Dynamic pressure, mm Wg (kg/m<sup>2</sup>)

$r$ : Density at sample point, kg/m<sup>3</sup>

If the inlet side is not ducted, then the velocity can be obtained directly by using an Anemometer.

Source: BEE, India

# Fan & Blower performance testing

Flow,  $Q \text{ m}^3/\text{s} = A \times V$

$A =$  cross sectional area ( $\text{m}^2$ ),  $\frac{\pi \times \text{Duct diameter}^2}{4}$

or height x width

$V =$  Velocity ( $\text{m/s}$ )

Air Horse power,  $\text{kW} = \frac{Q \times \Delta P_t \times g}{1000}$

$Q$ : Flow in  $\text{m}^3 / \text{sec}$

$H$ : Head in  $\text{m}$

$\Delta P_t$ : Total pressure rise in  $\text{mm wg}$  ( $\text{kg}/\text{m}^2$ )

$g$ : Acceleration due to gravity,  $9.81 \text{m}/\text{s}^2$

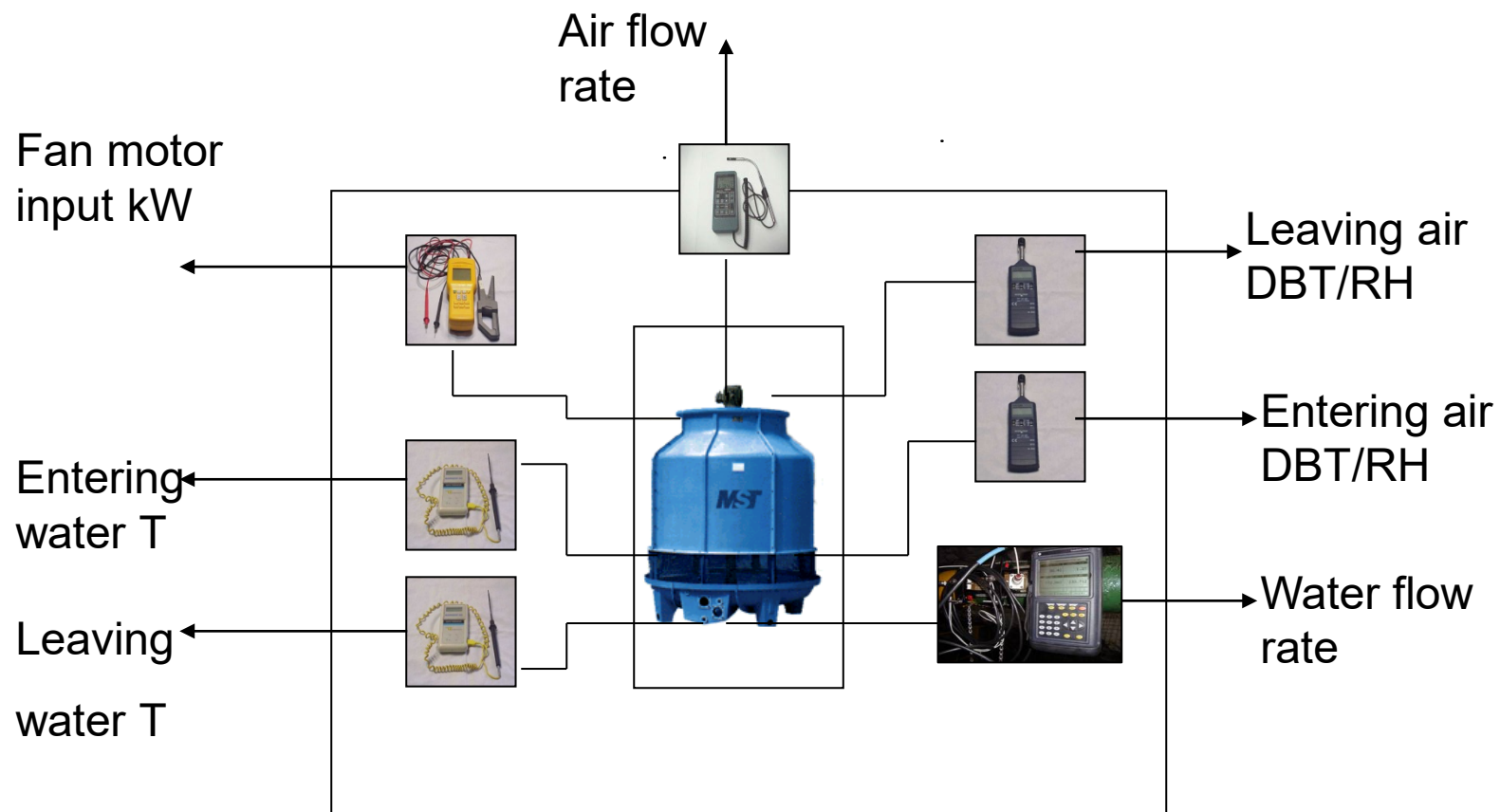
Shaft Horse Power (SHP),  $\text{kW}$   
 $=$  Motor input power,  $\text{kW} \times \eta_{\text{motor}}$

( $\eta_{\text{motor}}$  = Efficiency of motor)

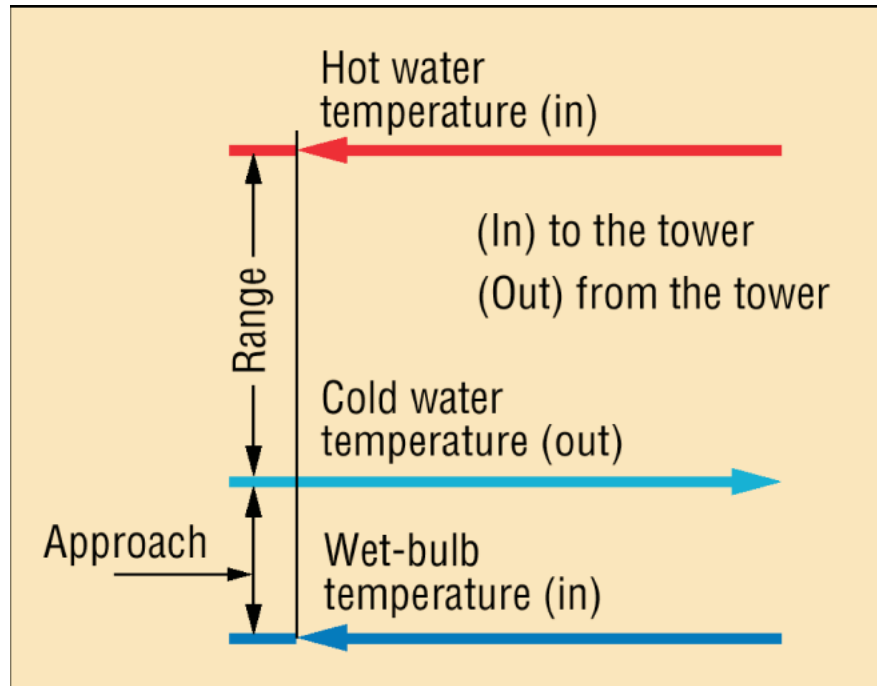
$$\text{Fan efficiency } \eta = \frac{AHP}{SHP}$$

Source: BEE, India

# Cooling tower performance testing



# Cooling tower performance testing



$$\text{Range} = T(\text{hot}) - T(\text{cold})$$

$$\text{Approach} = T(\text{cold}) - T(\text{wet bulb})$$

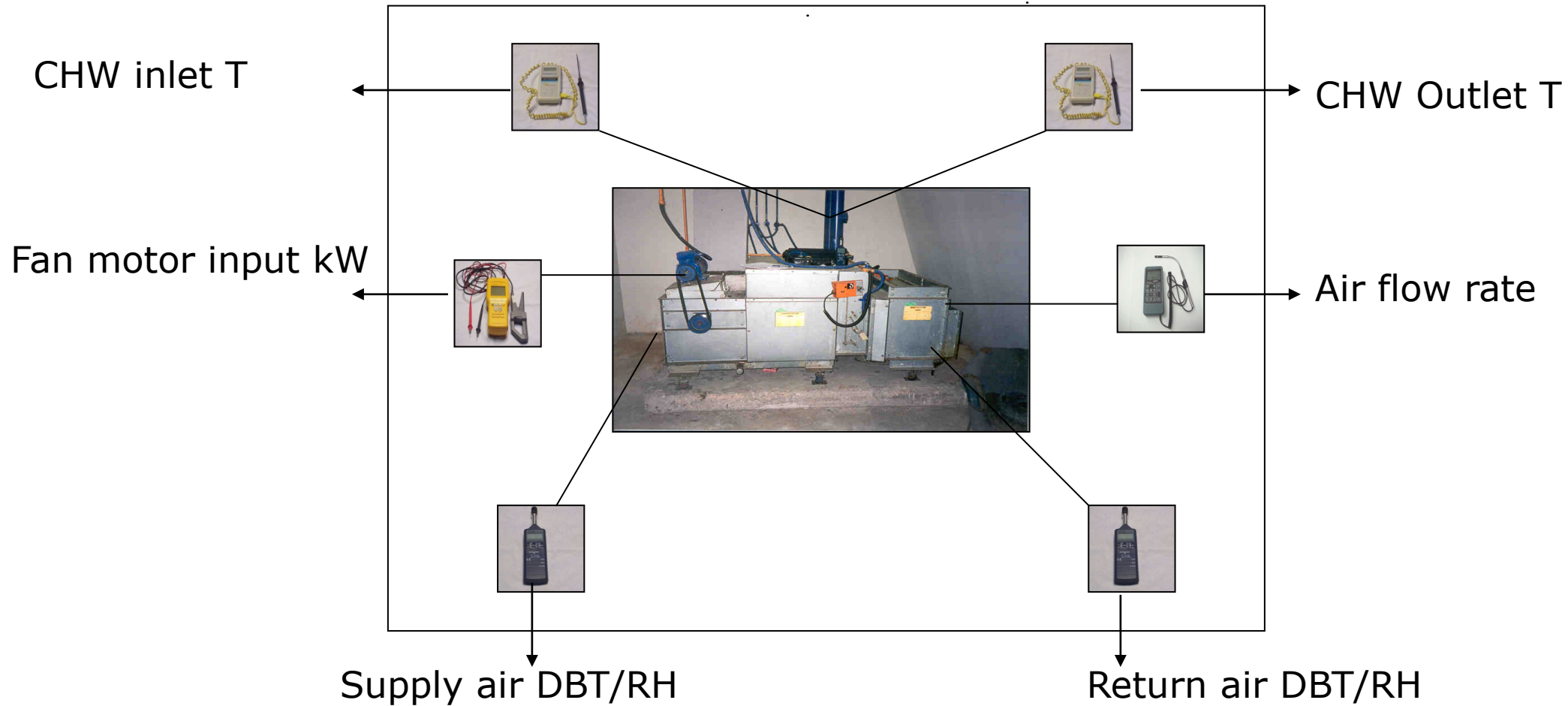
$$\text{Effectiveness} = \frac{\text{Range}}{\text{Range} + \text{Approach}}$$

$$\text{Evaporation loss} \left( \frac{\text{m}^3}{\text{hr}} \right)$$

$$= 0.00085 \times 1.8 \times \text{circulation rate} \left( \frac{\text{m}^3}{\text{hr}} \right) \times (T_{in} - T_{out})$$

Source: BEE, India

# AHU Performance testing



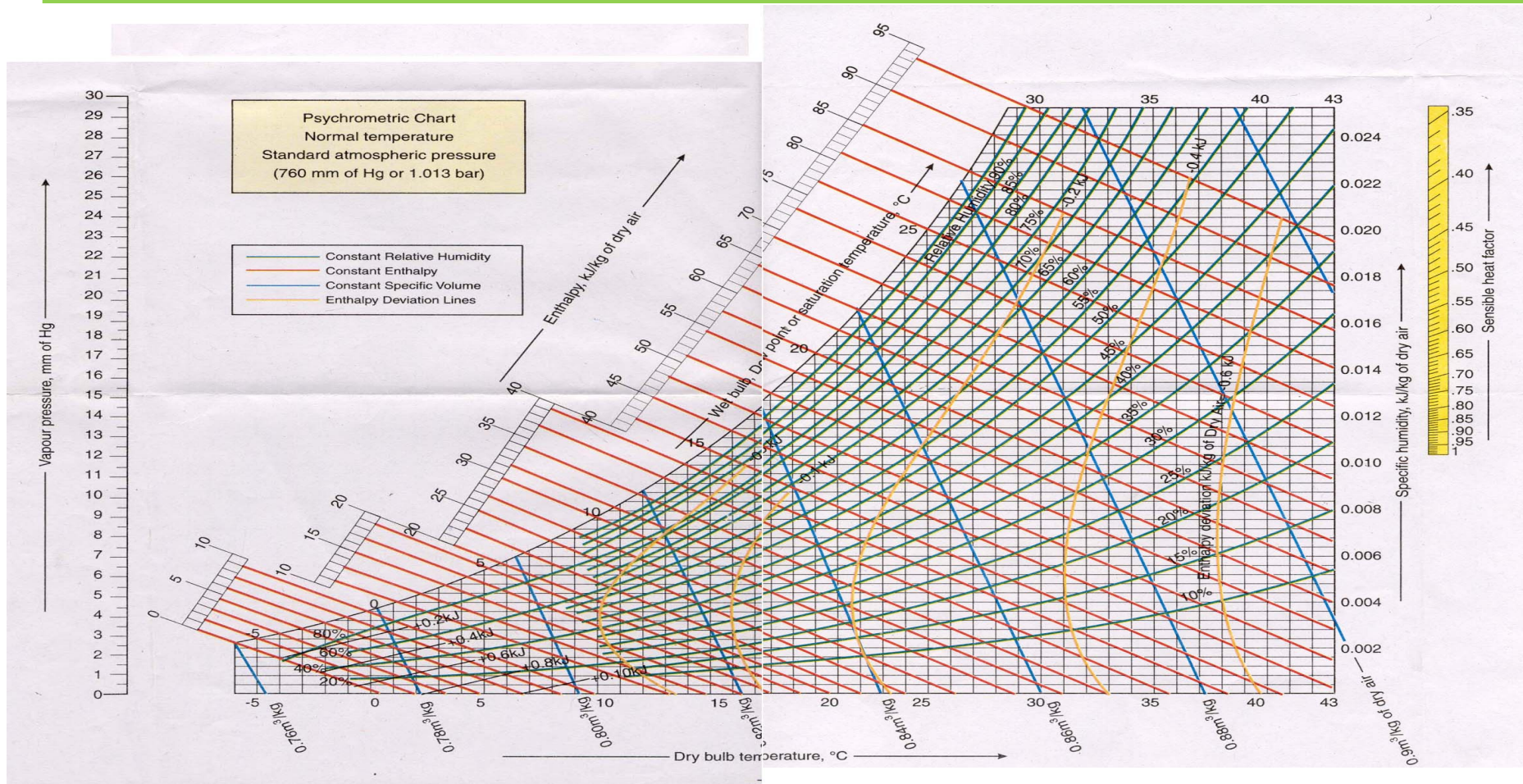
# AHU / package / DX unit performance testing

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Inlet air flow	: 21665 m <sup>3</sup> /h (6.02 m <sup>3</sup> /s)
Entering air	: 24.2°DBT, 17.2 °C WBT, 51.5% RH
Entering air enthalpy	: 52 kJ/kg
Leaving air	: 14.0°DBT, 12.5 °C WBT, 85% RH
Leaving air enthalpy	: 38 kJ/kg
Entering air density	: 1.05 kg/m <sup>3</sup>
Tons of refrigeration	: $21665 \times 1.05 \times \frac{(52-38)}{(3024 \times 4.18)} = 25.2 TR$



# AHU / package / DX unit performance testing



# Heat pump performance

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$$COP = \frac{Q \text{ useful heat}}{Q \text{ electric}}$$

If a heat pump releases 4 kW of heat and consumes 1 kW of electric power then

$$COP = \frac{4kW}{1 kW} = 4.0$$

# Boiler performance testing

$$\text{Boiler efficiency}(\eta) = \frac{\text{Heat output}}{\text{Heat input}} \times 100$$

$$= \frac{\text{Heat in steam output (kcal)}}{\text{Heat in fuel input (kcal)}} \times 100$$

$$= \frac{M_s \times (H_s - H_f)}{Q_c \times \text{GCV}} \times 100$$

## Example:

$$\frac{6000 \frac{\text{kg}}{\text{h}} \times (662 - 32) \frac{\text{kcal}}{\text{kg}}}{1200 \frac{\text{kg}}{\text{h}} \times 4930 \frac{\text{kcal}}{\text{kg}}} \times 100 = 63.9\%$$

## Parameters to be monitored:

- Quantity of steam generated per hour (**Ms**) in kg/h
- Quantity of fuel used per hour (**Qc**) in kg/h
- Working pressure (in kg/cm<sup>2</sup>(g)) and steam temperature (° C), corresponding enthalpy (**Hs**) in Kcal/kg
- The temperature of feed water (° C) and corresponding enthalpy (**Hf**) in kcal/kg
- Type of fuel and gross calorific value of the fuel (**GCV**) in kcal/kg of fuel .

# How to *save* energy?

# EE measures – air conditioning and refrigeration

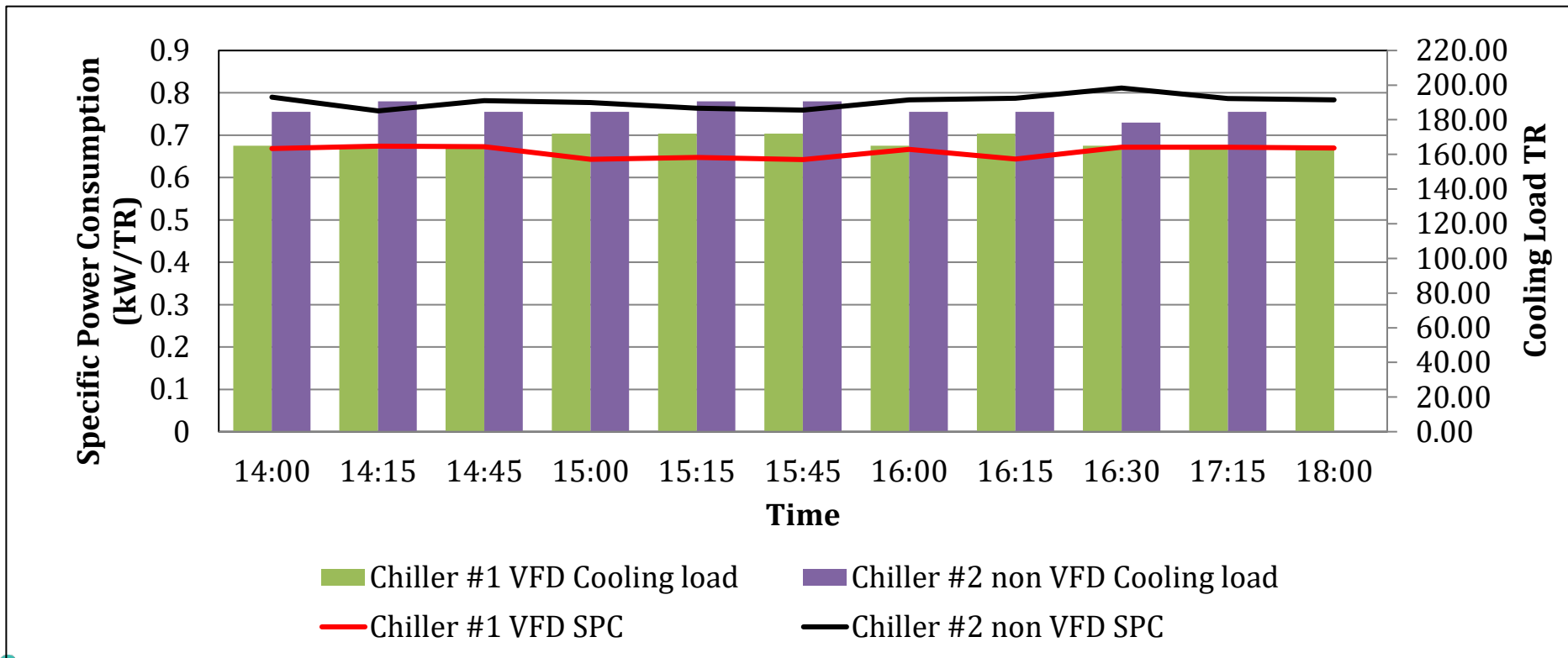
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- Follow best energy labelling standards available
- User of inverter integrated products.
- Maintenance of heat exchanger surfaces (evaporator, condenser, cooling coils)
- Matching capacity to system load
- Capacity control
- Chilled water storage



# Case: Chiller operations with VFD

## Specific Power Consumption (SPC) and Cooling load details comparison of Chiller #1 (VFD) and Chiller #2 (non-VFD)



**Annual energy savings with VFD : 2.2 Lakh kWh**



# EE measures – air conditioning and refrigeration

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- Economic insulation thickness - cold insulation
- Adoption of roof coatings/false ceiling/sun control films
- Adoption of variable air volume (VAV's)/air curtains
- Adoption of pre-cooling fresh air / optimum no. of air changes
- Reducing heat loads in conditioned space
- Heat recovery units and recuperators



# EE measures – air conditioning and refrigeration

## Capacity control types for vapour compression systems

Parameters	Reciprocating	Centrifugal	Screw
Capacity controls	▪ on/off (small)	▪ Inlet guide vane (IGV)	▪ Slide valve
	▪ Unloading of cylinders	▪ Variable speed drive with IGV	▪ Variable speed drive ▪ Suction throttling
Typical COP at part load upto 50%	Reduces at part load	Reduces at part load	Improves by 15 – 20%

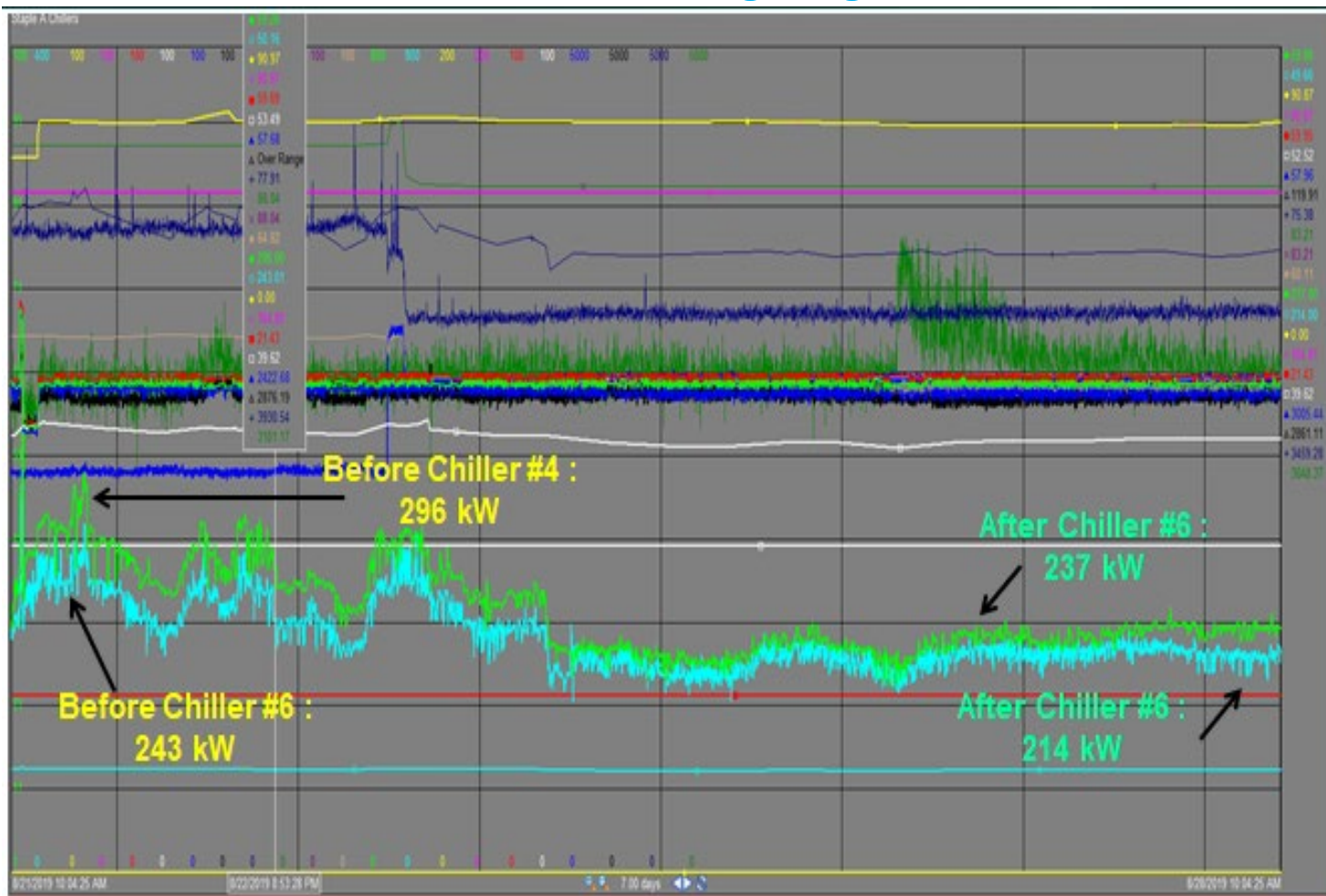


# EE measures – air conditioning and refrigeration

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- Least count of thermocouples to be used - 0.1°C
- In general,
  - chilled water flow - 0.68 m<sup>3</sup>/h per TR (3 gpm/TR)
  - Condenser water flow - 0.91 m<sup>3</sup>/h per TR (4 gpm/TR)
- A reduction of 0.55°C temperature in water returning from the cooling tower reduces compressor power consumption by 3%.
- A 1°C raise in evaporator temperature can help to save almost 3% on power consumption
- Lowest possible cooling tower water should be passed through chillers
- Leaving water temperature (Evaporator) should be monitored (most of the cases)

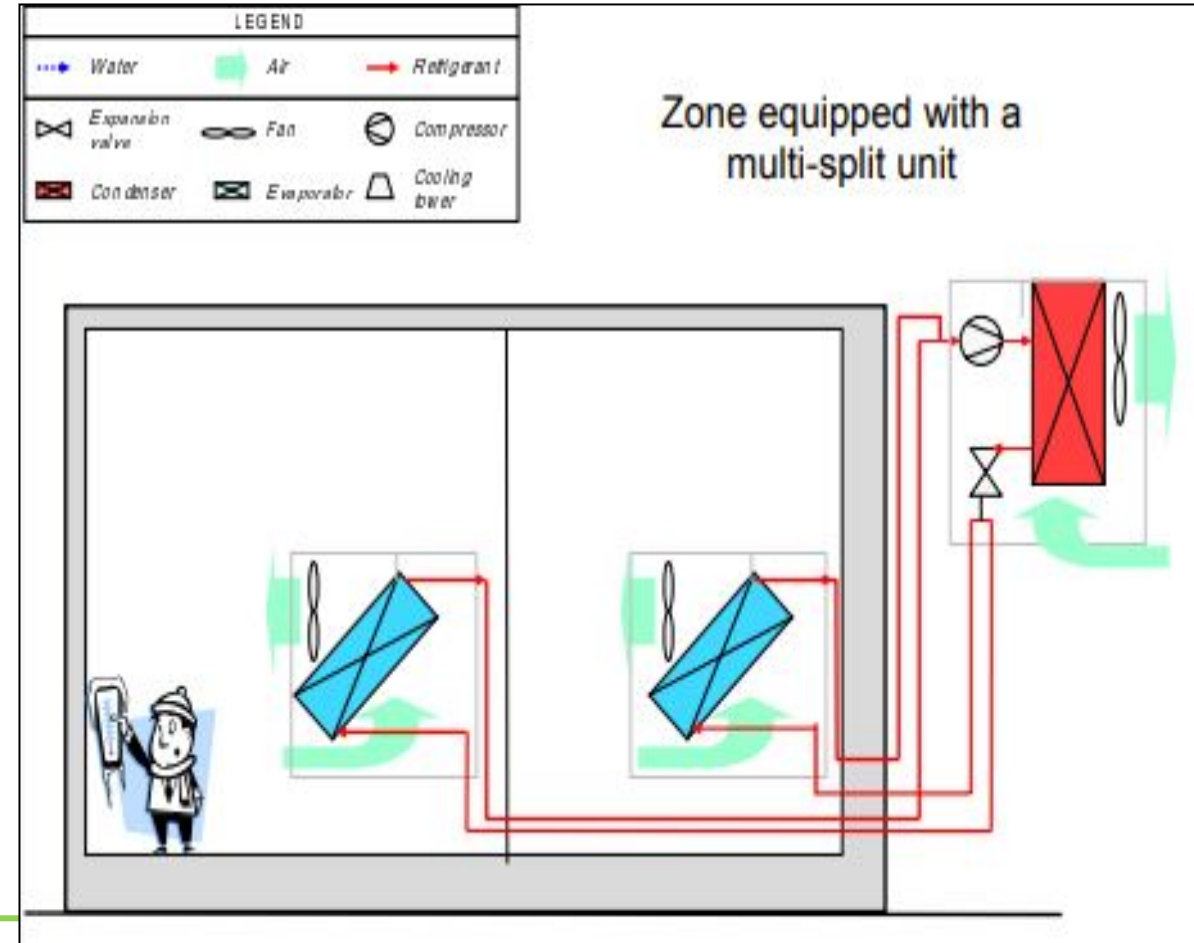
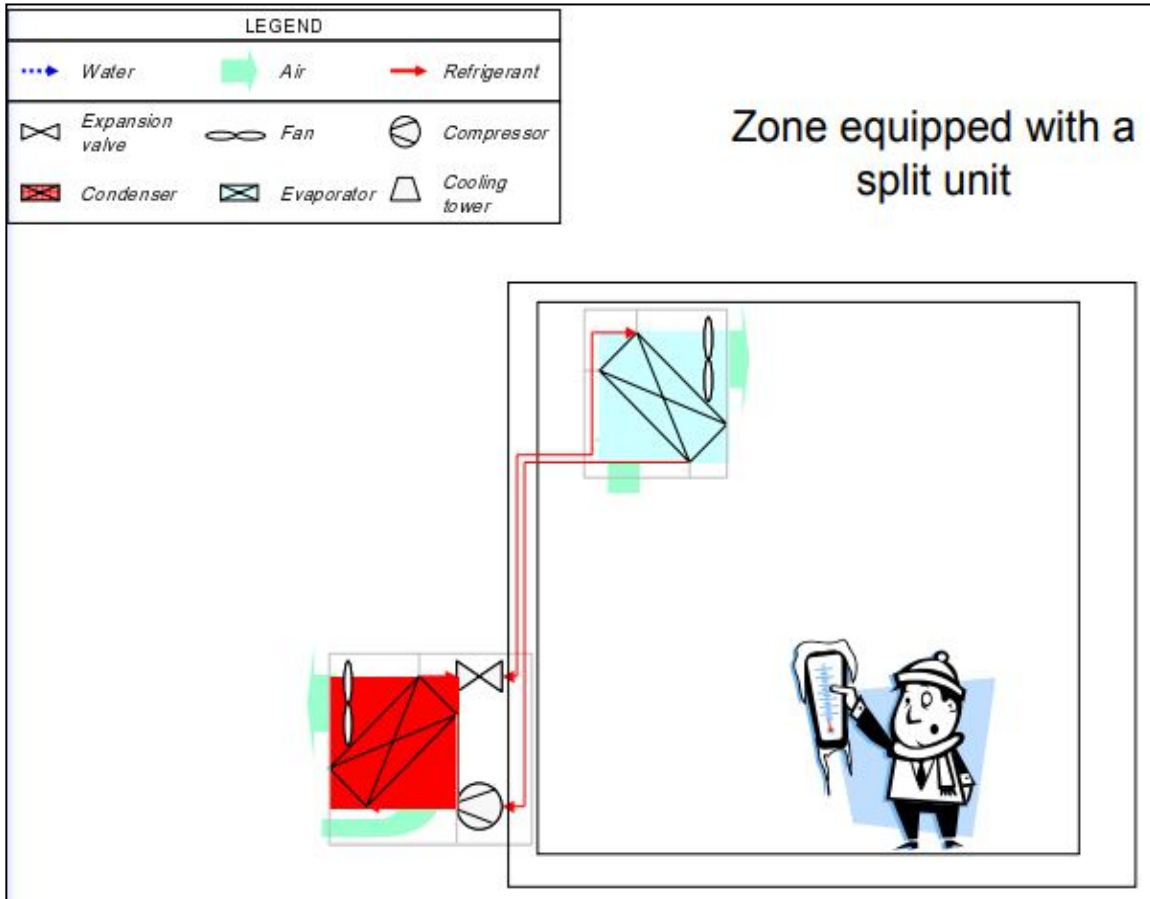
# Case: Rectify system flows and pressures



Particulars	Unit	Before	After
<b>Chiller #4</b>			
SPC	kW/TR	0.617	0.444
COP	kW/kW	5.69	7.9
<b>Chiller #6</b>			
SPC	kW/TR	0.533	0.459
COP	kW/kW	6.59	7.65

**Annual energy savings achieved : 6.3 Lakh kWh**

# EE measures – air conditioning and refrigeration



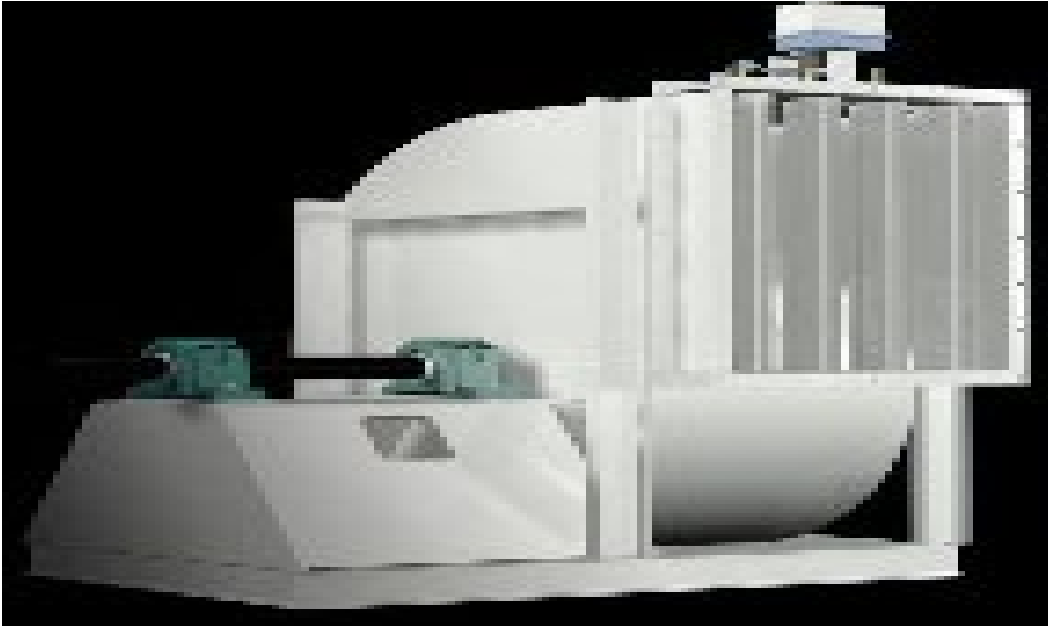
# EE measures – Pumps, Fans and blowers

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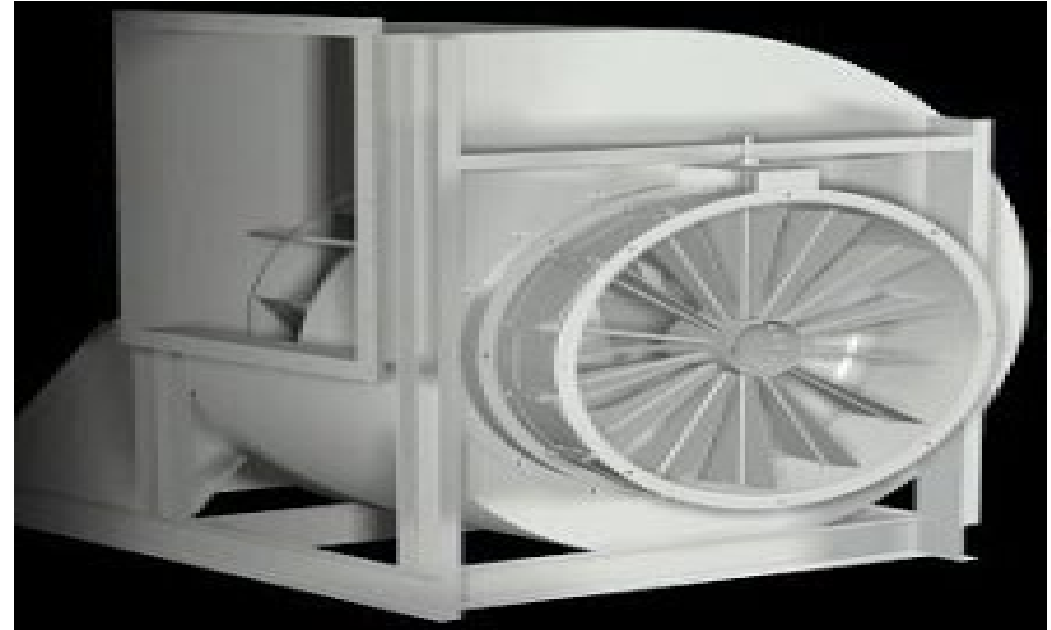
- Avoid oversizing
- Change of impeller by high efficiency impeller
- Replacing with high efficiency equipment
- Impeller de-rating (by a smaller dia impeller)
- Speed reduction by pulley dia change (for fans and blowers)
- Options for energy efficient flat belts in place of V-belts (for fans and blowers)
- Adopting Inlet guide vanes in place of damper control (for fans and blowers)
- Minimizing system resistance and pressure drops
- Variable speed drive / variable speed fluid coupling application

# EE measures – Fans and blowers

## Damper change



## Inlet Guide Vanes



## Control mechanisms

## EE measures – Fans and blowers

<b>Centrifugal Fans</b>	<b>Peak Efficiency Range</b>
<i>Airfoil, backward curved/inclined</i>	79-83
<i>Modified radial</i>	72-79
<i>Radial</i>	69-75
<i>Pressure blower</i>	58-68
<i>Forward curved</i>	60-65
<b>Axial fan</b>	
<i>Vanaxial</i>	78-85
<i>Tubeaxial</i>	67-72
<i>Propeller</i>	45-50



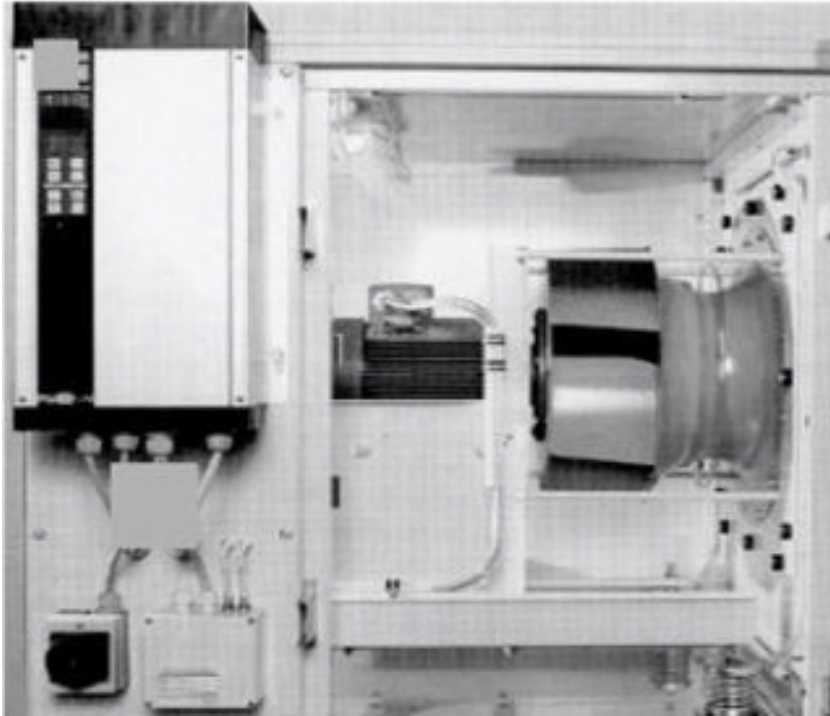
## Case : Air loss through stand-by blowers



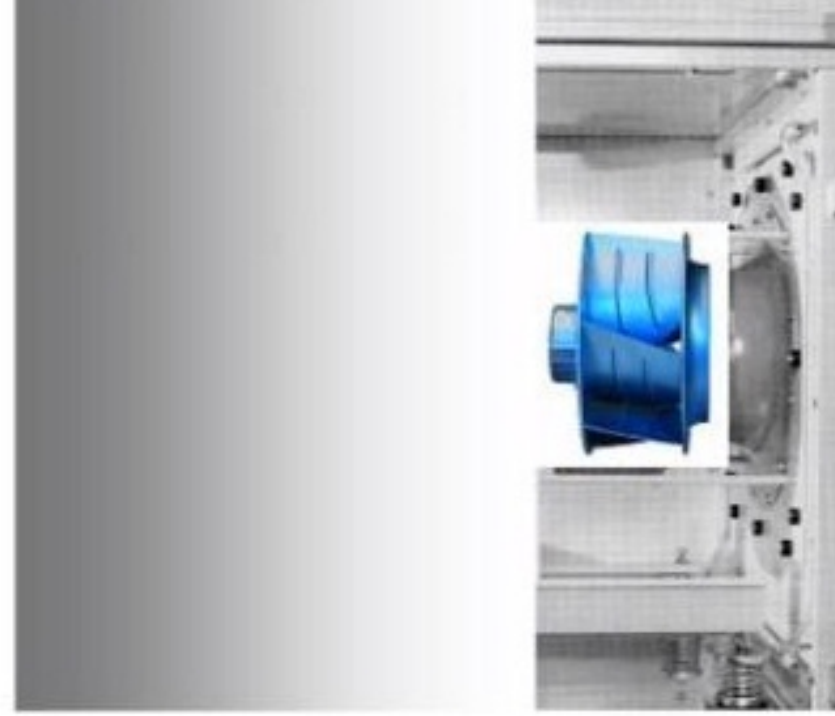
**Annual energy  
Savings:  
0.21 lakh kWh**

**Payback -  
Immediate**

## Case 7: Technology upgradation - AHU



Conventional motor-fan with VFD



EC motors

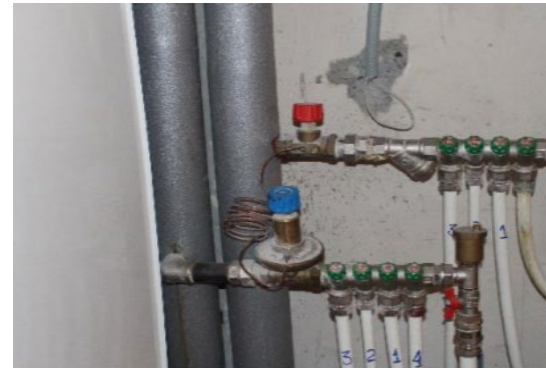


## Case 6: Technology upgradation - AHU

S No.	AHU Name			Actual Before		Actual After		Saving
		Rating KW	After Rating KW	KW	CFM	KW	CFM	
1	Ahu-4	15	12	9.38	7791	4.64	7810	51%
2	Ahu-1	15	12	9.22	11456	5.06	11520	45%
3	Ahu-8	7.5	6	5.96	5456	2.36	5490	60%
4	AHU 2	11	15	9.5	23150	5.31	23985	44%
5	AHU 3	7.5	6.8	6.22	18207	4	18613	36%
6	AHU 5	15	15	12.81	27968	8.5	25580	34%
7	AHU 6	15	15	15.01	18681	9.3	18735	38%
8	AHU 7	11	12	5.52	8132	2.75	8875	50%
9	AHU 9	11	10.2	7.06	8213	3.58	8522	49%
10	QA 12	11	10	10.09	9667	4.69	10894	54%
11	Warehouse-G	15	15	12.06	17555	6.34	17980	47%
12	Warehouse-F	15	15	13.1	15993	6.86	16025	48%

# EE measures - Heating

- Avoid electric resistance heaters
- If electricity is the only choice, heat pumps are preferable in most climates - they easily cut electricity use by 50% when compared with electric resistance heating
- Insulation of pipes and equipment
- Installation of thermostatic regulators on radiators
- Install balancing valves
- Energy metering and monitoring



Source: [https://www.unece.org/fileadmin/DAM/energy/se/pdfs/geee/study/Final\\_Master\\_file\\_-\\_March\\_11\\_final\\_submission.pdf](https://www.unece.org/fileadmin/DAM/energy/se/pdfs/geee/study/Final_Master_file_-_March_11_final_submission.pdf)

# EE measures - Heating

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## Boilers

- Optimal stack temperatures
- Encourage feed water preheating (economizer, solar thermal or heat pump integration)
- Avoid incomplete combustion
- Excess air control
- Minimize surface radiation and convection losses
- Use of condensing boilers

Source: [https://www.unece.org/fileadmin/DAM/energy/se/pdfs/geee/study/Final\\_Master\\_file\\_-\\_March\\_11\\_final\\_submission.pdf](https://www.unece.org/fileadmin/DAM/energy/se/pdfs/geee/study/Final_Master_file_-_March_11_final_submission.pdf)

# EE measures - Heating

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## Boilers (residential and commercial buildings)

<b>Old low efficiency</b>	<b>Mid-efficiency</b>	<b>High – efficiency</b>
Natural draft	Exhaust fan controls for combustion air and gases	Condensing flue gases in second heat exchanger for extra efficiency
Heavy heat exchanger	Electronic ignition	Sealed combustion
56% - 70% AFUE	Compact and light weight	90% to 98.5% AFUE
	Small dia flue pipe	
	80%-83% AFUE	

Source: <https://www.energy.gov/energysaver/home-heating-systems/furnaces-and-boilers>

# Case: Use of Heat pumps

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## Background :

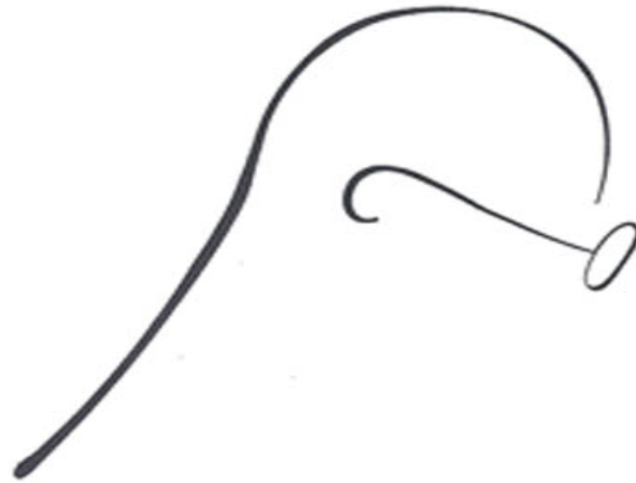
- Diesel fired boilers used for hot water generation in a hotel
- City's location weather highly suitable for heat pump installation.
- The diesel fuel high cost also supported the change.

## Recommendation :

- Heat pumps to be installed.
- Existing boiler may be used for back up

## Savings:

**13.5 kL annual diesel savings**



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# Thank You

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**Thank you for your attention**

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